

The Islamic University
Technical Engineering College
Building & Construction Tech. Eng. Dept.

أستاذ المادة : دكتور علي العذاري



2021-2022

الجامعة الإسلامية
كلية الهندسة التقنية
قسم هندسة تقنيات البناء والإنشاءات

محاضرات مقاومه المواد/الجزء الأول
الجزء الثاني(اسئله محلولة وواجبات) يوزع اثناء المحاضرات

2021-2022



strength of material

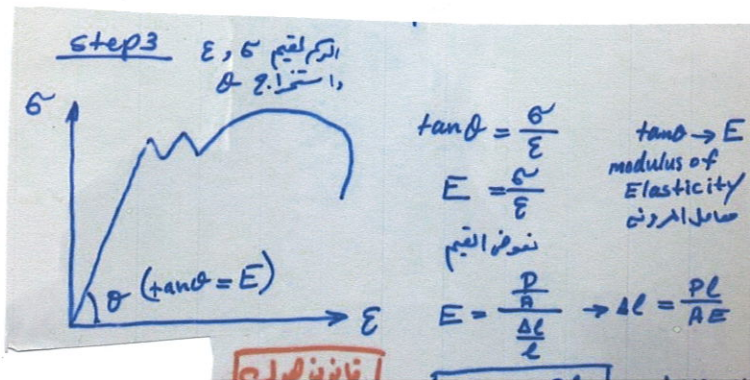
مقاومه المواد

(2021-2022)

الصف الثاني/بناء وانشات

القسم الاول

ملاحظه : القسم الثاني(التمارين والواجبات) يوزع في اوقات المحاضرات



1-10-2021
د. علي الغزالي

-1-

معا ومما الحوار
الصف الثاني / بغداد و اشبانات

Stress & Strain

الاجود الانتقال

I

Stresses : Tension & Compression

stress: σ : force per unit Area القوة في وحدة المساحة

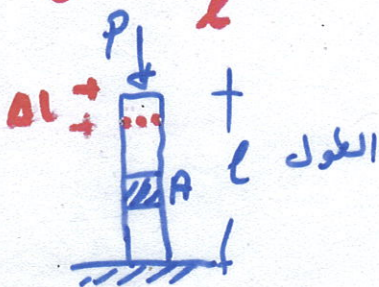
Strain: ϵ : Elongation per unit length الاستطالة في وحدة الطول

$$\sigma = \frac{P}{A}$$

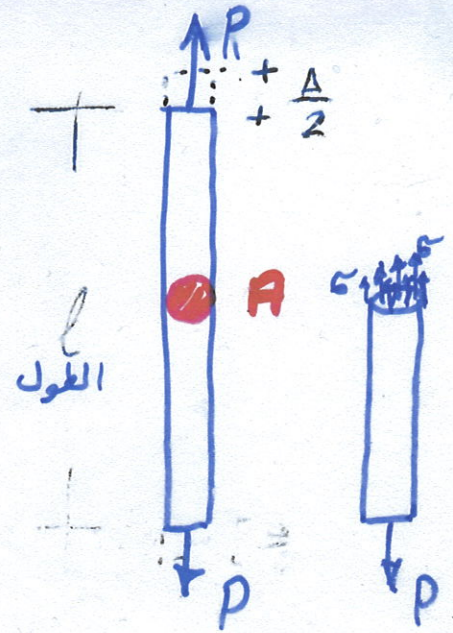
[N/mm^2] الوحدات

$$\epsilon = \frac{\Delta L}{L}$$

[-] الوحدات



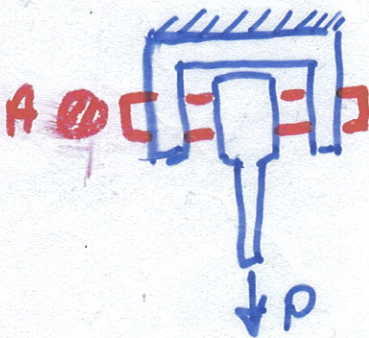
Compression (Contraction)



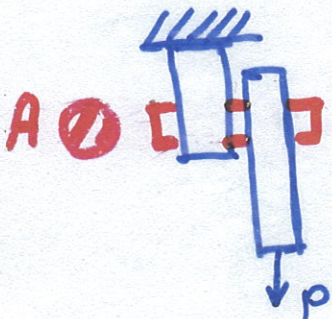
Tension (elongation)

Stresses

Shearing Stress (τ)

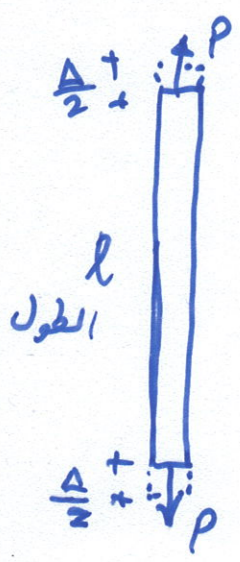
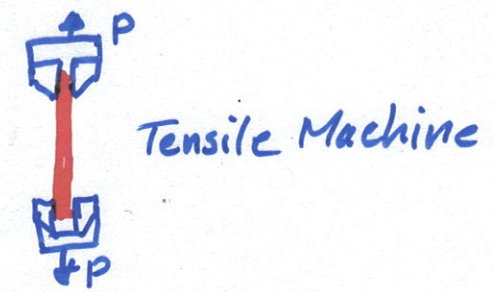


$$\tau = \frac{P}{2A} \quad [\frac{N}{mm^2}]$$



$$\tau = \frac{P}{A} \quad [\frac{N}{mm^2}]$$

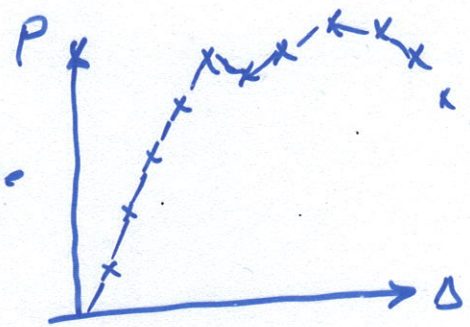
Tensile Test فحص الشد



Step 1

P [N]	Δ [mm]
...	...
...	...
...	...
...	...
...	...
...	...

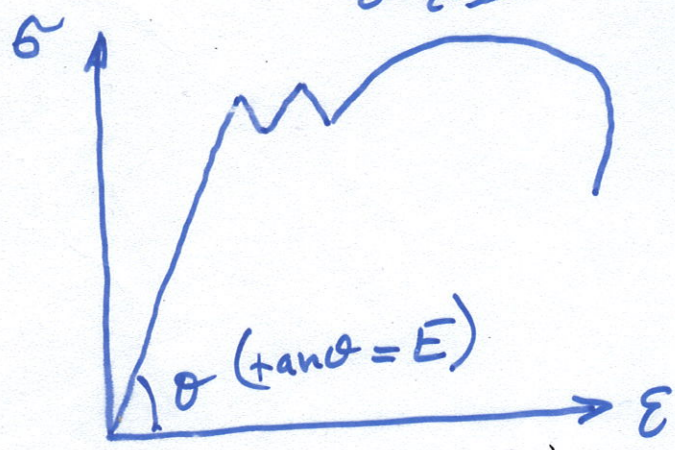
Curve \rightarrow



Step 2 **نقسم على التوابت A و l**

P/A	Δ/l	$\sigma (\frac{P}{A})$	$\epsilon (\frac{\Delta}{l})$
...
...
...
...
...
...

Step 3 **التركيب قيم σ و ϵ واستخراج E**



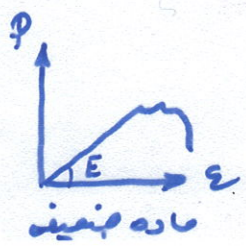
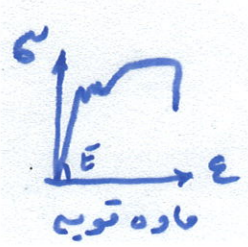
$\tan \theta = \frac{\sigma}{\epsilon}$
 $E = \frac{\sigma}{\epsilon}$
 modulus of Elasticity
 معامل المرونة

$E = \frac{P/A}{\Delta l / l} \rightarrow \Delta l = \frac{Pl}{AE}$

قانون هوك
 $\epsilon = \frac{\sigma}{E}$

$\Delta l = \frac{Pl}{AE}$

$E = \text{Constant for every Material}$
 ثابت لكل مادة



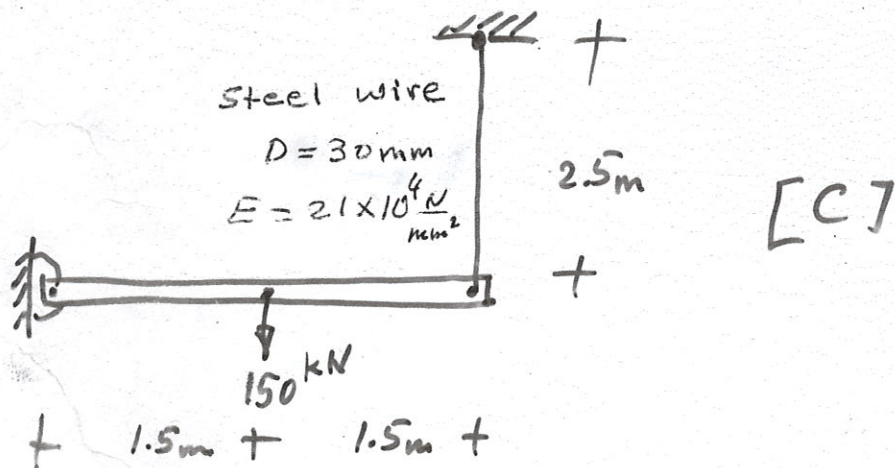
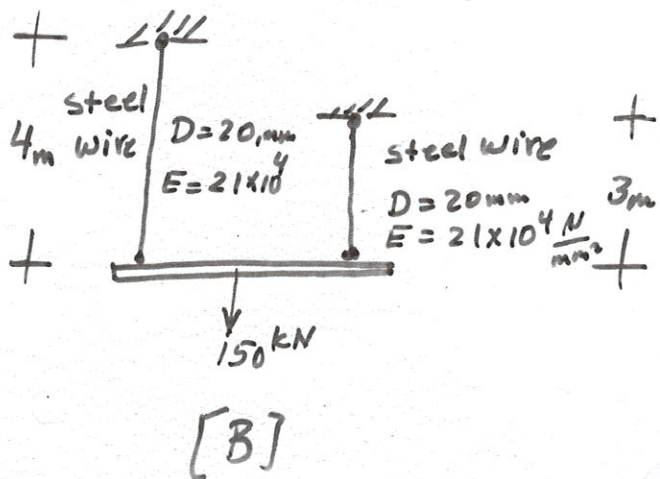
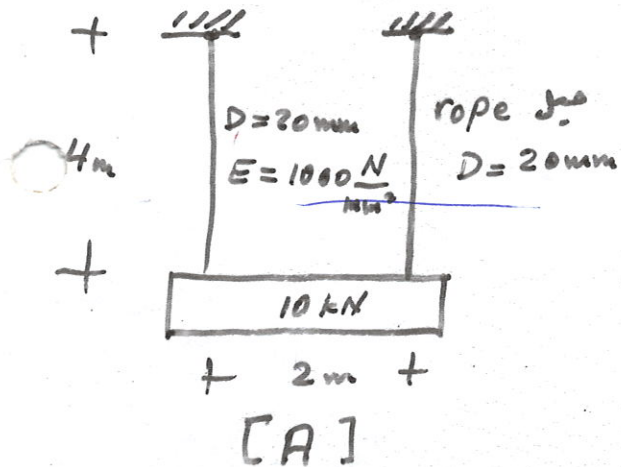
الاستطالة

Exercises تمارين
stress & strain

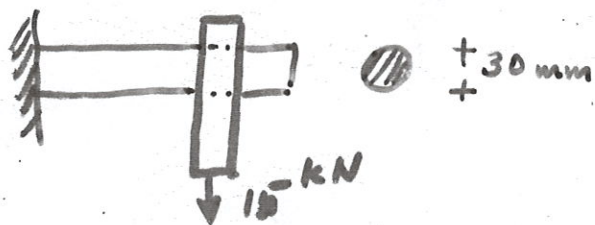
1. A steel Bar of dimension $1 \times 6 \text{ mm}$ and length $l = 30 \text{ m}$
Calculate the stress and Elongation for a Force 50 N
knowing that $E = 200 \times 10^3 \frac{\text{N}}{\text{mm}^2}$

Asw.: $\sigma = 8.34 \frac{\text{N}}{\text{mm}^2}$, $\Delta = 1.25 \text{ m}$

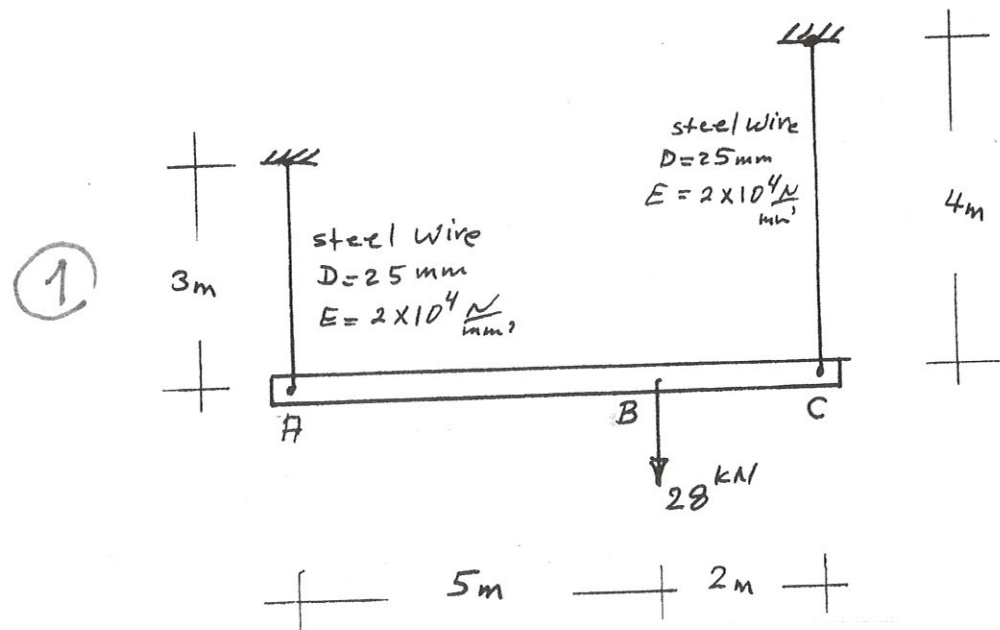
2. Find the stresses and Elongation



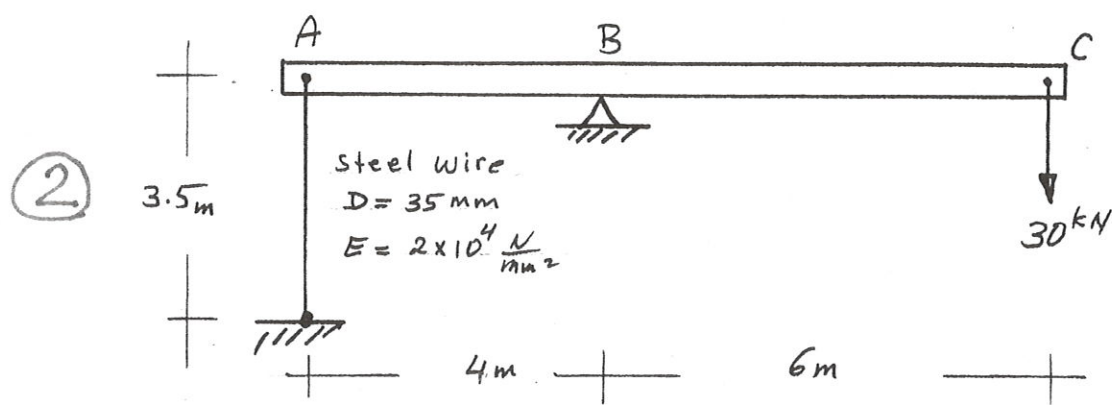
3. Calculate the shearing stresses in the Circular bar



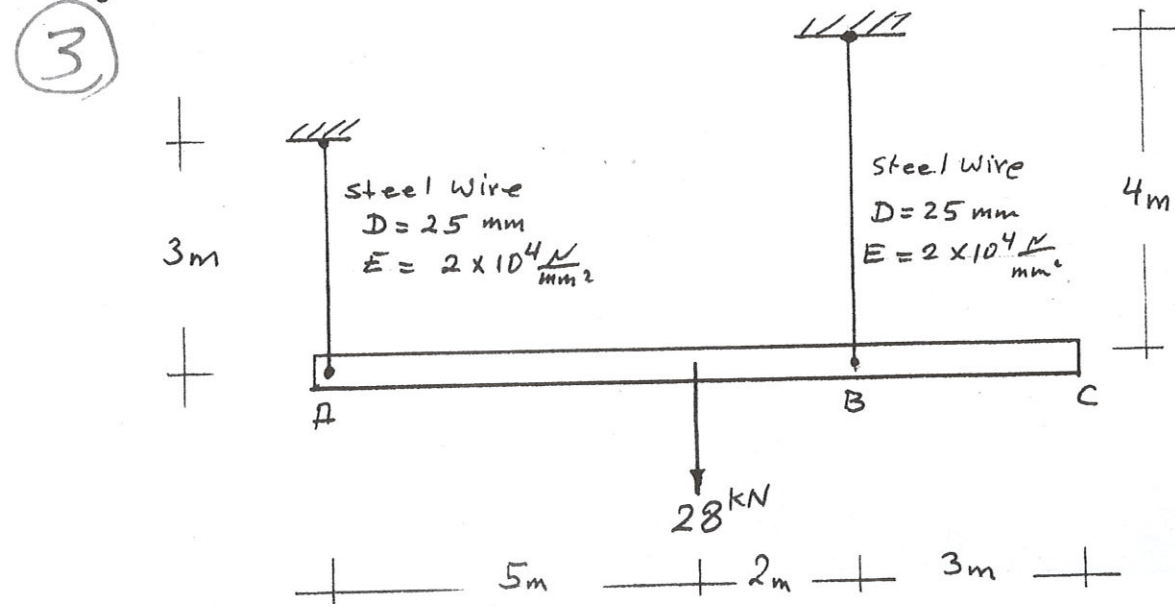
Q₁ - Calculate the vertical movement of points A, B and C



Q₂ - Calculate the vertical movement for points A, B and C



Q₃ - Calculate the vertical movement of points A, B and C



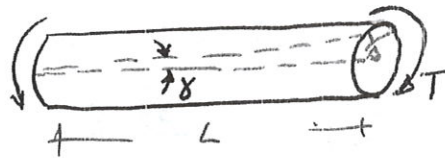
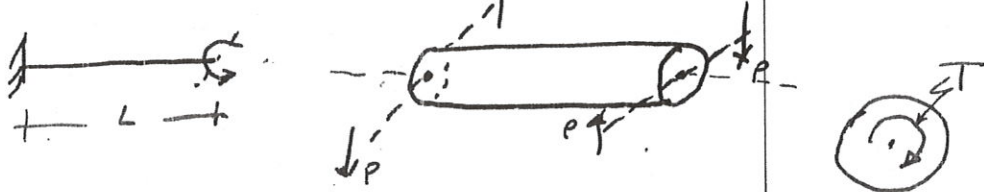


المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د. علي العزازي

20-4-2016
 د. علي العزازي

الصف: ثمان
 1 مقاوم المواد

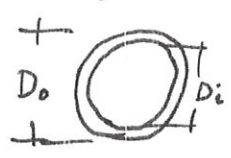
Torsion (اللي)



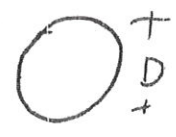
Torsional stress : $\tau = \frac{T r}{J}$

J: polar moment of Inertia

Polar moment of Inertia



$J = \frac{\pi}{32} (D_o^4 - D_i^4)$

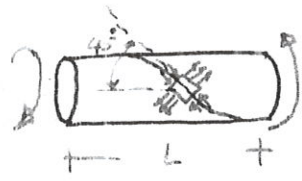
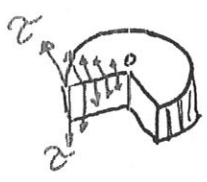


$J = \frac{\pi D^4}{32}$

Torsional strain γ
 modulus of shear G

$G = \frac{\tau}{\gamma}$ $\frac{\text{Stress}}{\text{Strain}}$

angle of twist $\theta = \frac{T L}{G J}$

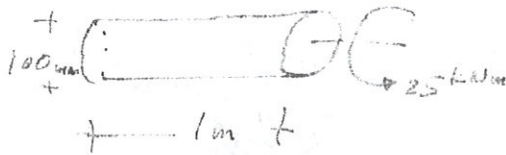




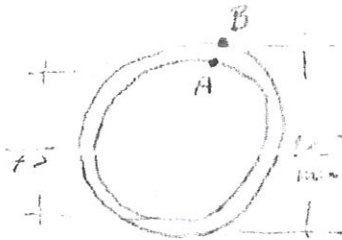
(12) 3

وايب

Q₁ - Calculate the Angle of twist θ for the bar having
 $G = 85 \times 10^3 \frac{N}{mm^2}$



Q₂ - If we have the inside shearing stress (point A) $\tau = 50 \frac{N}{mm^2}$
 Calculate the Outside shearing stress (point B) i.e. τ_{max}



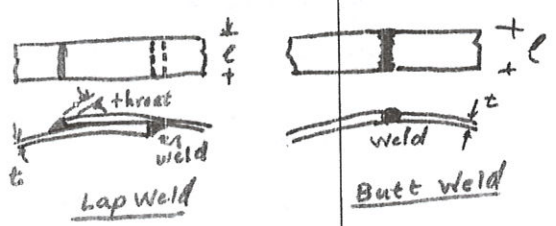


- 2 -

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Welded Joints

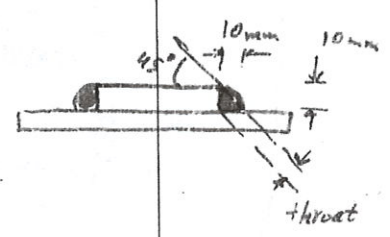
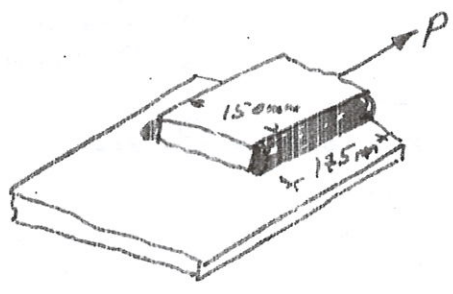
الربط: جفاف الحام



$$\sigma_{(Weld)} = 112 \frac{N}{mm^2} \text{ in Tension \& Comp}$$

$$\tau_{(Weld)} = 95 \frac{N}{mm^2} \text{ in Shear}$$

Example 2: Calculate the allowable Force P using the allowable shearing stress = $80 \frac{N}{mm^2}$



(Answ: P = 193 kN)



المرحلة: الثانية

السنة الدراسية: 2017-2018

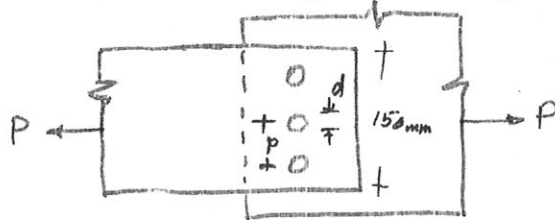
اسم التدريسي: أ.م.د علي العذاري

10-11-2015
علي العذاري

واجب / امتحان
Rivet and welding joints

(16) مقاومه المواد
الصف الثاني

Q₁ - Find the strength of the joint using:



$$t = 6 \text{ mm}$$

$$\phi = 12 \text{ mm}$$

$$p = 50 \text{ mm}$$

$$\text{allowable } \sigma_{\text{tensile}} = 120 \frac{\text{N}}{\text{mm}^2}$$

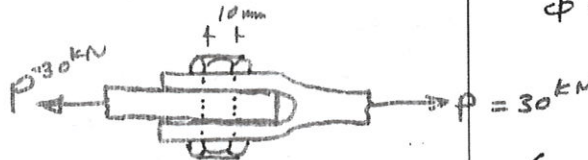
$$\sigma_{\text{Bearing}} = 180 \%$$

$$\tau_{\text{shear}} = 90 \%$$

(Answ: 30.54 kN)

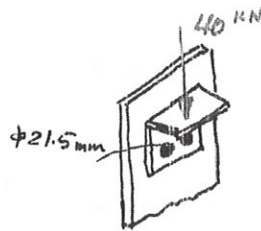
Q₂ - Calculate the stresses in the Rivet using: $P = 30 \text{ kN}$

$$\phi = 10 \text{ mm}$$



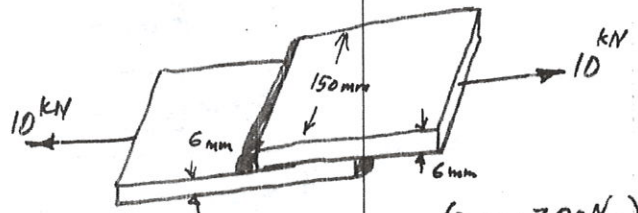
(Answ: 191 $\frac{\text{N}}{\text{mm}^2}$)

Q₃ - Calculate the shearing stress in the Rivet:



(Answ: 55 $\frac{\text{N}}{\text{mm}^2}$)

Q₄ - Calculate the stress in the weld



(Answ: 7.86 $\frac{\text{N}}{\text{mm}^2}$)



المرحلة: الثانية

السنة الدراسية: 2017-2018

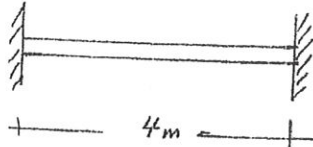
اسم التدريسي: أم.د. علي العذاري

2015-12-20
د.ع.م. (العذاري)

2
نماذج أسئلة امتحانية سابقة
الإجرات الحرارية والانفعالات

18
مقاومة المواد
الثاني
Thermal stresses and strain

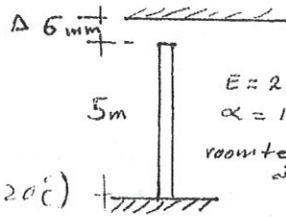
Q1- calculate the force and stress acting on the beam when the temperature raise from 5 c to 75 c



$E = 2 \times 10^5 \frac{N}{mm^2}$, $\alpha = 12 \times 10^{-6} \frac{1}{C}$, $A = 300 mm^2$
 t from 5° to 75°

(Answer: $P = 0.504 kN$)
 $\sigma = 1.76 \frac{N}{mm^2}$

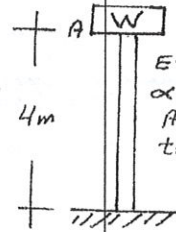
Q2- calculate the raise of temperature in order to close the Gape (6mm) for the column:



$E = 2 \times 10^5 \frac{N}{mm^2}$
 $\alpha = 12 \times 10^{-6} \frac{1}{C}$
room temperature = 20°C

(Answer: $t_2 = 120°C$)

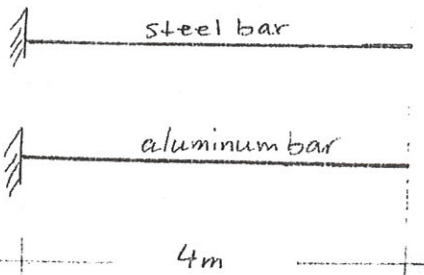
Q3-a) calculate the weight W in order to compel point A to stay in it's position when the temperature raise from 5 c to 75 c
b) what happen to point A when we remove the weight W with the same raise in temperature?



$E = 2 \times 10^5$
 $\alpha = 12 \times 10^{-6}$
 $A = 300 mm^2$
 t from 5° to 75°

(Answer: a) 0.54 kN
b) 3.36 mm)

Q4- Calculate elongation difference between the 2-bars shown below when the temperature raise from 30 to 120°C



steel: $E = 2 \times 10^5 \frac{N}{mm^2}$
 $\alpha = 12 \times 10^{-6} \frac{1}{C}$

aluminium: $E = 0.5 \times 10^5 \frac{N}{mm^2}$
 $\alpha = 23 \times 10^{-6} \frac{1}{C}$

(Answer: $\Delta = 3.96 mm$)

$\Delta L_1 - \Delta L_2$



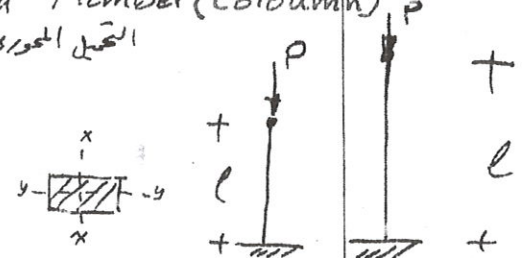
2017/14 - 2018/15
 د. علي العذاري

- 1 -
 مقادير المواد

(29)

المصف الثاني / صيني

Axially loaded Member (Column)
 التحميل المحوري للعمود

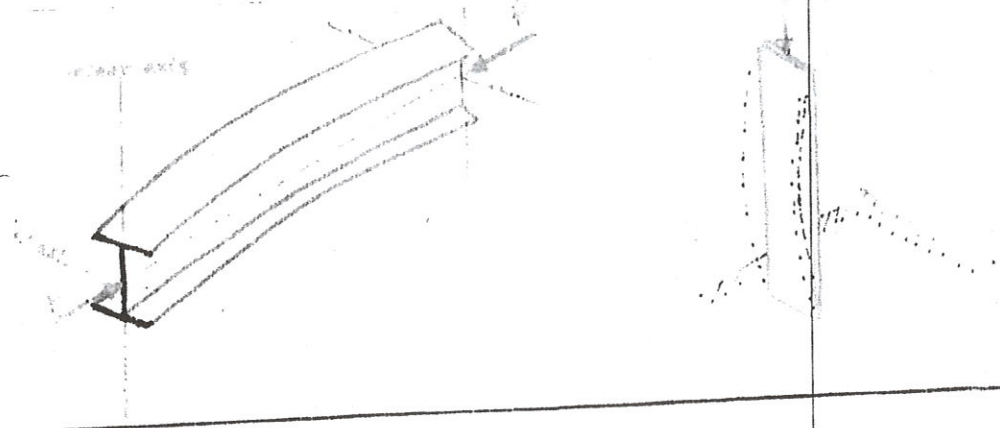


Types of Column: → 1) Short Column 2) Long Column

slenderness ratio = $\frac{\text{effective length}}{\text{Least radius of gyration}}$

$\lambda = \frac{le}{r_y}$

failure { Short Column (compression failure) = σ_c
 Long Column (Buckling or Bending failure) = σ_c

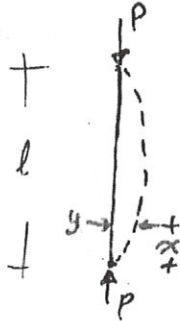


References:

- 1) Structure, theory and analysis by Williams & Todd, USA 2000
- 2) Theory of elastic stability by Timoshenko & Gere, USA 1961
- 3) The stability of frames by Merchant & Horne, England 1965
- 4) Strength of Material, by G. K. HIRASKAR, Khanna publishers, Delhi, India



-2-
Euler's Formula for long columns:
(Buckling Load) (30)



$$M_x = EI \frac{d^2y}{dx^2} = -Py$$

$$\frac{d^2y}{dx^2} + \frac{Py}{EI} = 0$$

solving the differential equation for P gives:

the least Buckling load

$$P_E = \frac{\pi^2 EI}{l^2}$$

for the stress at Buckling σ_c :

$$P_E = \frac{\pi^2 EI}{l^2} \quad , \quad \sigma_c = \frac{P}{A} \rightarrow P = \sigma_c A$$

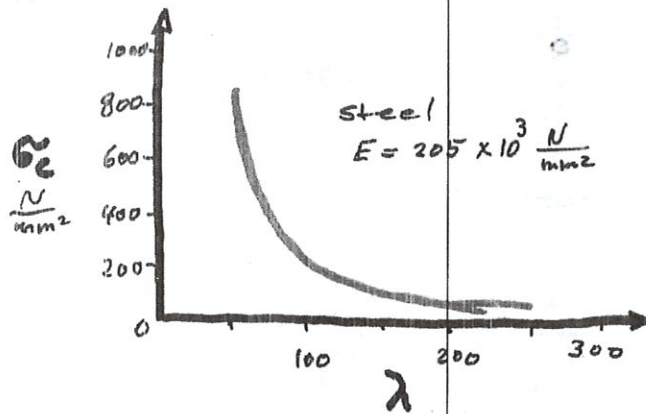
$$\sigma_c A = \frac{\pi^2 EI}{l^2} \left(\frac{A}{A} \right) \rightarrow \text{use radius of gyration:}$$

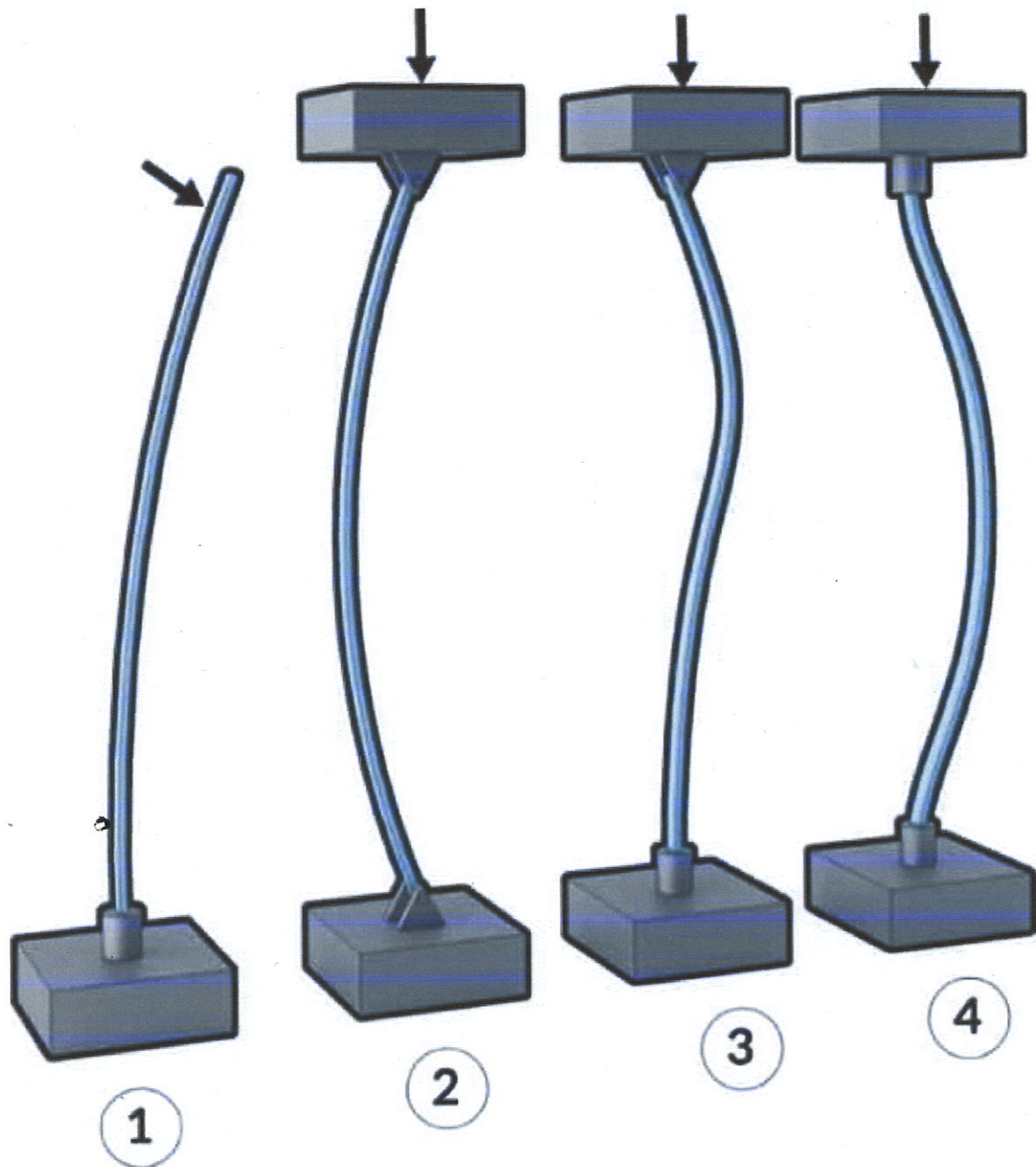
$$r^2 = \frac{I}{A}$$

$$\sigma_c = \frac{\pi^2 E}{l^2} \cdot r^2 \rightarrow \text{use Slenderness ratio:}$$

$$\lambda = \frac{l}{r}$$

$$\sigma_c = \frac{\pi^2 E}{\lambda^2}$$





المرحلة: الثانية

السنة الدراسية: 2017-2018

اسم التدريسي: أ.م.د علي العذاري



الكلية الإسلامية الجامعة
قسم هندسة تقنيات البناء والإنشاءات
المادة: مقاومة مواد

-4-

Example 1:

A steel W254 x 89 Column shown with

$$A = 11400 \text{ mm}^2$$

$$I = 48.49 \times 10^6 \text{ mm}^4, E = 205 \times 10^3 \frac{\text{N}}{\text{mm}^2}$$

Calculate the greatest length of Column that can be used without buckling.

Solution:

$$\sigma = \frac{P}{A} = \frac{1600 \times 10^3}{11400} = 140.4 \text{ MPa}$$

$$\sigma_c = \pi^2 \frac{E}{\lambda^2} \quad \lambda = \frac{l_c}{r_y}$$

$$\text{slenderness ratio} \rightarrow \lambda = \sqrt{\frac{\pi^2 E}{\sigma_c}} = \pi \sqrt{\frac{205 \times 10^3}{140.4}} = 120$$

$$r_y = \sqrt{\frac{I_y}{A}} = \sqrt{\frac{48.49 \times 10^6}{11400}} = 65.2 \text{ mm}$$

for fixed-pinned member $l_c = 0.7l$, the

slenderness ratio can be written as

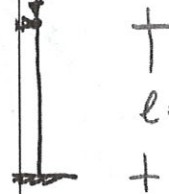
$$\lambda = \frac{l_c}{r_y} = \frac{0.7l}{65.2}$$

$$120 = \frac{0.7l}{65.2}$$

$$l = 11177 \text{ mm}$$

$$l = 11.18 \text{ m}$$

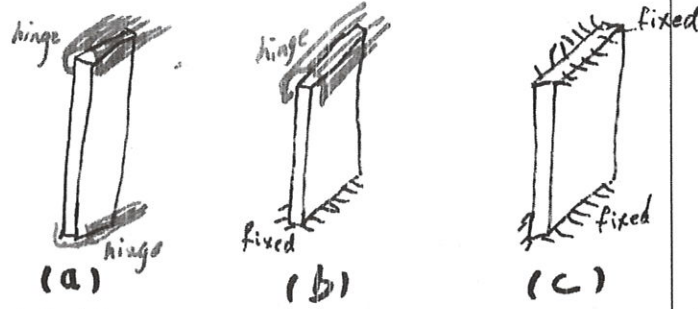
P = 1600 kN



(32)



- 6 -



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Solution

a) $P_{c_e} = \frac{\pi^2 EI_x}{l^2} = 70.16 \text{ kN}$

$P_{c_e} = \frac{\pi^2 EI_y}{l^2} = 31.192 \text{ kN} \checkmark$

$\sigma = \frac{P}{A} = \frac{31.192}{40 \times 60} = 13 \text{ MPa} \left(\frac{N}{mm^2} \right)$

$r = \sqrt{\frac{I}{A}} = \sqrt{\frac{32 \times 10^4}{40 \times 60}} = 11.54$

$\lambda = \frac{le}{r_y} = \frac{2 \times 4500}{11.54} = 779$

b) $P_{c_x} = 143 \text{ kN} \rightarrow \left[2.04 \frac{\pi^2 EI_x}{l^2} = 2.04 \times 70.16 = 143 \right]$

$P_{c_y} = 63 \rightarrow \left[2.04 \frac{\pi^2 EI_y}{l^2} = 2.04 \times 31.192 = 63 \right]$

$\sigma = 26.25 \rightarrow \frac{63}{40 \times 60} = 26.25 \frac{N}{mm^2}$

$r = 11.54$

$\lambda = \frac{0.7 \times 4500}{11.54} = 272$

c) $P_{c_x} = 280 \text{ kN}$

$P_{c_y} = 124 \text{ kN}$

$\sigma = 52$

$r = 11.54$

$\lambda = \frac{0.5 \times 4500}{11.54} = 195$

من الخصائص
 $I_x = 72 \times 10^4 \text{ mm}^4$
 $I_y = 32 \times 10^4 \text{ mm}^4$



المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د. علي العذاري

ع.أ.ع. / ع.ع.ع. - ع.أ.ع.ع.
 علي العذاري

Axially loaded Member

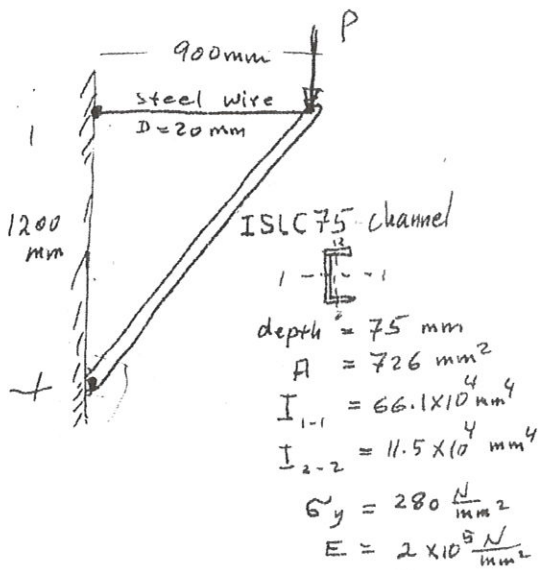
36 مقاومة المواد
 الصف الثاني

Q1 - A 180 cm length of ISWB150 standard I-Beam is to be as a pin-ended column. Calculate the critical value of the load
 (Ans: $P_{cr} = 576 \text{ kN}$)

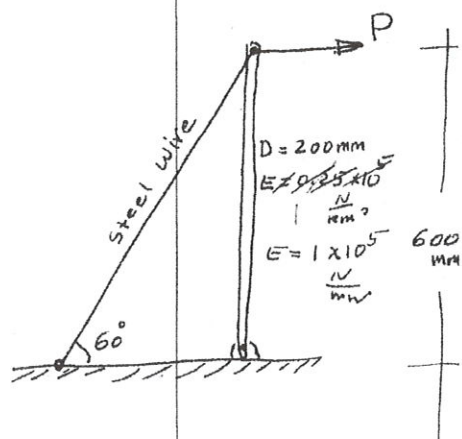
Q2 - An ISWB 550 steel Beam is to be used as a column 13.5m long with the lower end built-in and the upper end pinned. Calculate the max compressive load that the column can carry.
 (Ans: 825 kN)

Q3 - Find the largest value of the vertical load P that the structure can support?

Q4. Find the maximum value of the horizontal force P that can be applied at the top of the pole without causing collapse?



الجواب (Ans: $P_{max} = 80.63 \text{ kN}$)

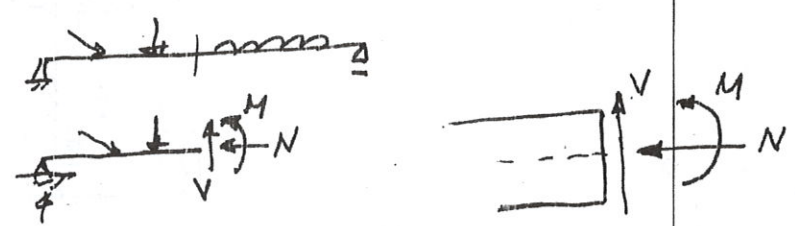
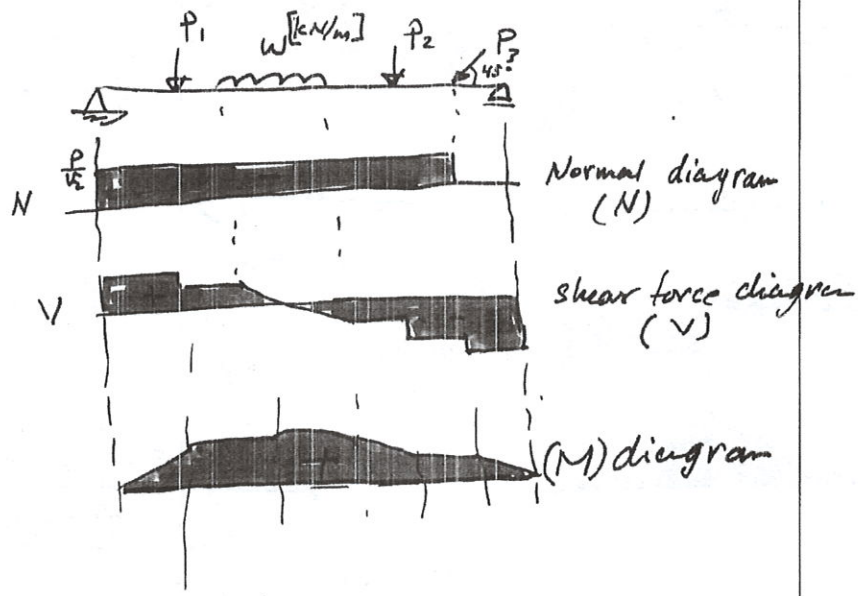


(Ans: $P = 124 \text{ kN}$) الجواب



المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د علي العذاري

مقاومة المواد - Strength of Mat. (مقاومة المواد)
 الخلية الإسلامية الجامعة - ابعث الأخرى
 مقامه المواد - (20)
 المصنف الثاني - قسم هندسة
 تقنيات البناء والإنشاءات
 Bending moment, Shearing force and Normal force DIAGRAM
 مخططات عزوم الاضمار وقوى القعر والقوى المحورية



Case 1

at \$x=0\$
 $V = \frac{wL}{2}$
 $M = 0$

at \$x = \frac{L}{2}\$
 $V = 0$
 $M = \frac{wL}{2} \cdot \frac{L}{2} - w \left(\frac{L}{2} \right)^2$
 $= \frac{wL^2}{4} - \frac{wL^2}{8}$

$N = \frac{wL^2}{8}$

The diagrams for Case 1 show a beam of length \$L\$ with a triangular load \$w\$ (KN/m). Below the beam are the shear force diagram (V) and the bending moment diagram (M). The shear force diagram is a straight line from \$wL/2\$ at \$x=0\$ to 0 at \$x=L/2\$. The bending moment diagram is a cubic curve starting at 0 at \$x=0\$ and reaching a maximum at \$x=L/2\$.

مصادر الخط المستقيم

$$V = \frac{wL}{2} - wx$$

$$M = \frac{wL}{2} \cdot x - w \cdot x \cdot \frac{x}{2}$$

$$M = \frac{wL}{2} x - w \frac{x^2}{2}$$

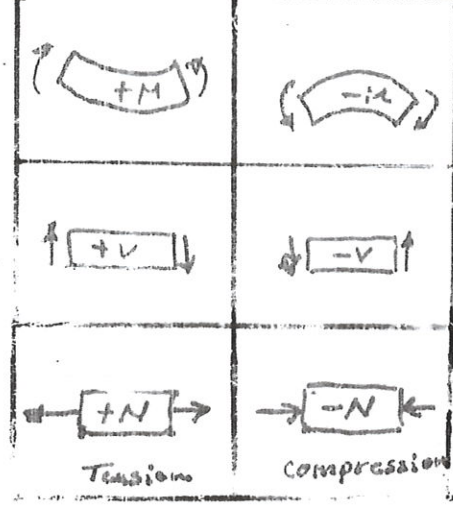
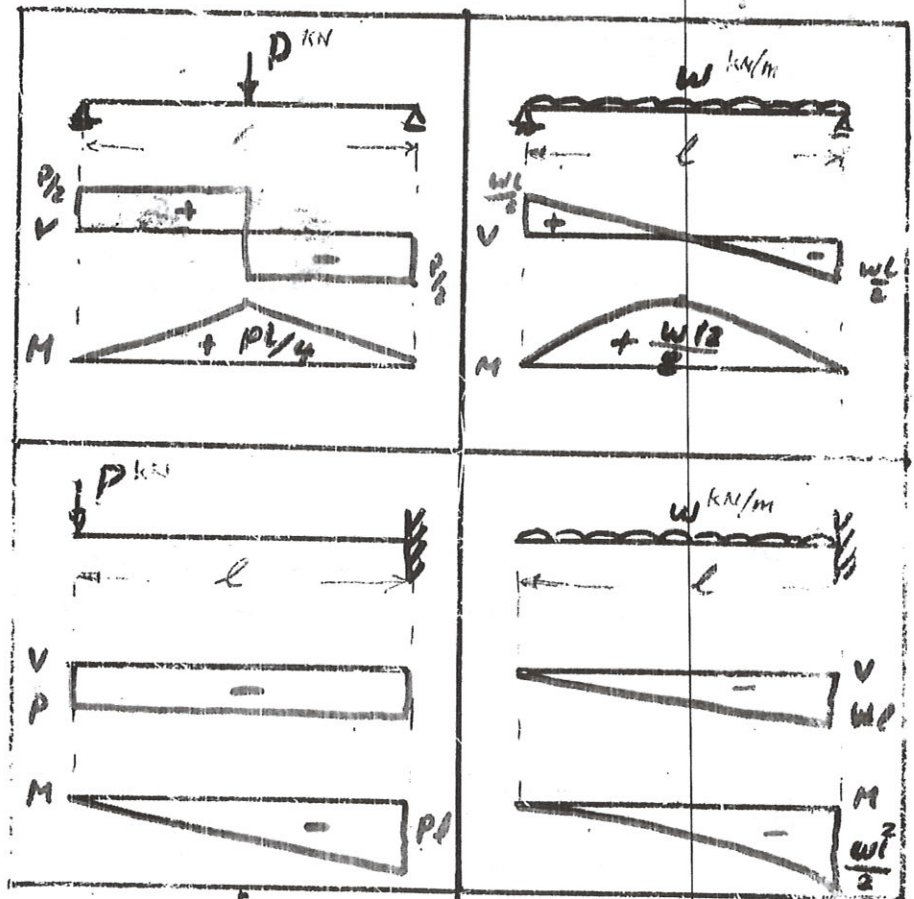
مصادر ال Parabola القطع المكافئ



المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د. علي العذاري

مقاومة المواد / المصف الثاني
 د. علي العذاري
 20-2-2016 Bending moment & Shearing Force & Normal force
 Diagrams
 مخططات عزز الإضواء وقوى الشد والقوى المحورية

(27) - 8 -



اتفاقية الاشارة
 Sign convention



المرحلة: الثانية

السنة الدراسية: 2017-2018

اسم التدريسي: أم.د. علي العذاري

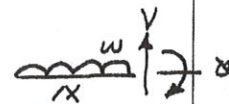
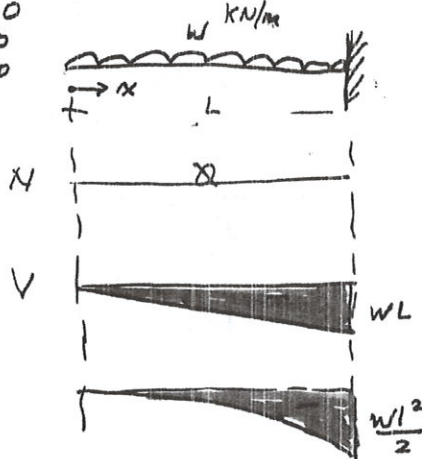
- 3 -

22

20-2-2018

Case 4

at $x=0$
 $V=0$
 $M=0$



خط مستقيم
خط منحنى M

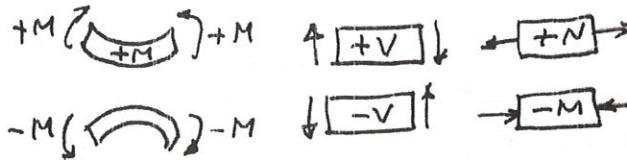
$$V = w \cdot x$$

$$M = w \cdot x \cdot \frac{x}{2} = \frac{w x^2}{2}$$

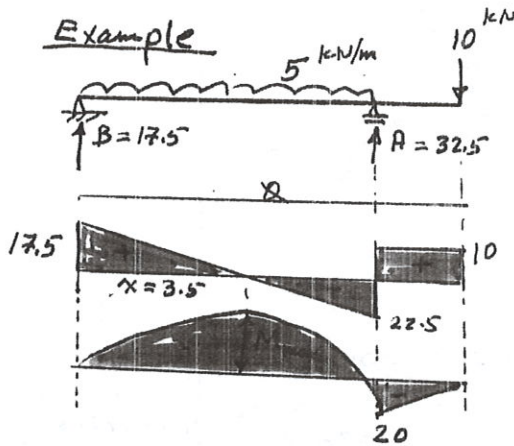
at $x=0$
 $V=0$
 $M=0$

at $x=L$
 $V = wL$
 $M = \frac{wL^2}{2}$

Sign convention or إتفاقيات الأعداد



Example



$$A \times 8 - 5 \times 8 \times 4 - 10 \times 10 = 0$$

$$A = \frac{260}{8} = 32.5 \text{ kN}$$

$$B \times 8 + 10 \times 2 - 5 \times 8 \times 4 = 0$$

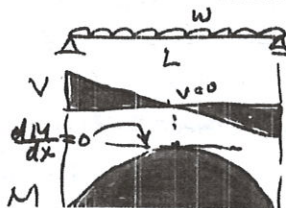
$$B = \frac{140}{8} = 17.5 \text{ kN}$$

M_{max} @ $V=0$

$$\frac{17.5}{x} = \frac{22.5}{8-x} \rightarrow x = 3.5 \text{ m}$$

$$M = 17.5 \times 3.5 - 5 \times 3.5 \times \frac{3.5}{2}$$

$$= 30.625 \text{ kNm}$$



$$M = \frac{wL}{2} \cdot x - w \frac{x^2}{2}, \quad V = \frac{wL}{2} - wx$$

$$\frac{dM}{dx} = \frac{wL}{2} - w \cdot 2 \frac{x}{2} = 0 \rightarrow \frac{wL}{2} - wx = 0 \quad x = \frac{L}{2}$$

ملاحظة: في الحد الأقصى يكون $V=0$ الفرق أكبر يجب عند نقطة $V=0$

$$\therefore M_{max} \text{ @ } V=0$$

المرحلة: الثانية

السنة الدراسية: 2017-2018

اسم التدريسي: أم. د. علي العذاري

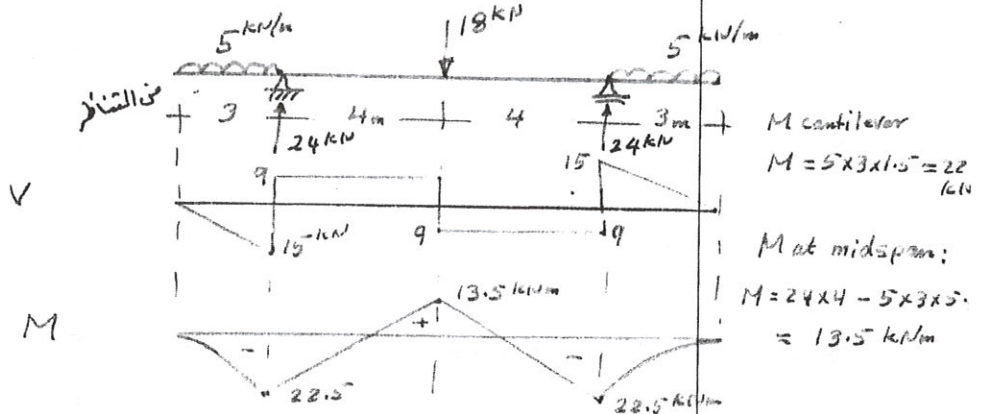


الكلية الإسلامية الجامعة
قسم هندسة تقنيات البناء والإنشاءات
لادة: مقاومة مواد

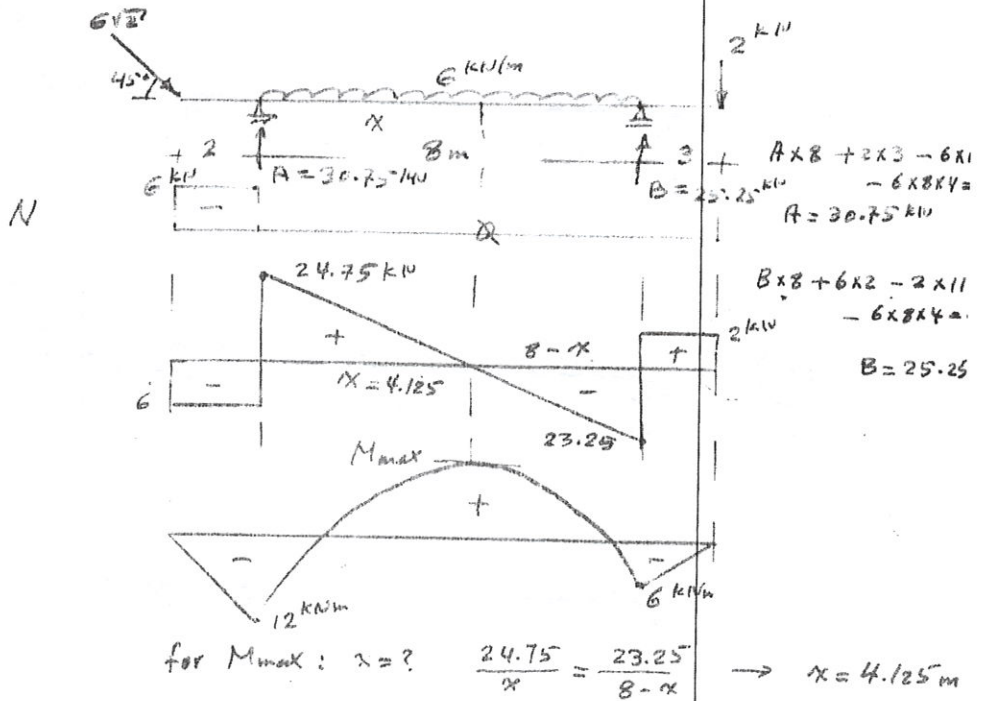
- 5 -

(24)

(4)



(5)



$$M_{\text{max}} = 30.75 \times 4.125 - 6 \times 4.125 \times \frac{4.125}{2} - 6 \times 6.125 = 39.05 \text{ kNm}$$



المرحلة: الثانية

السنة الدراسية: 2017-2018

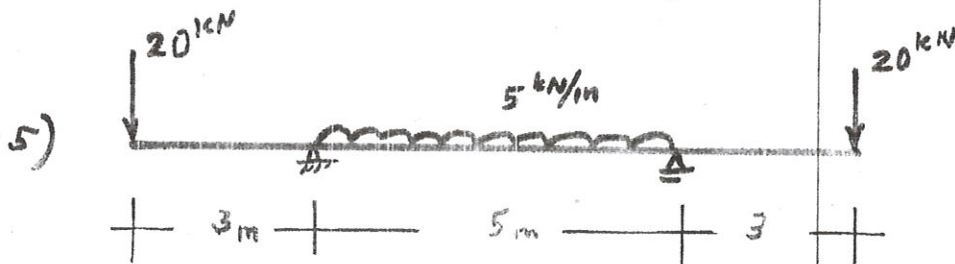
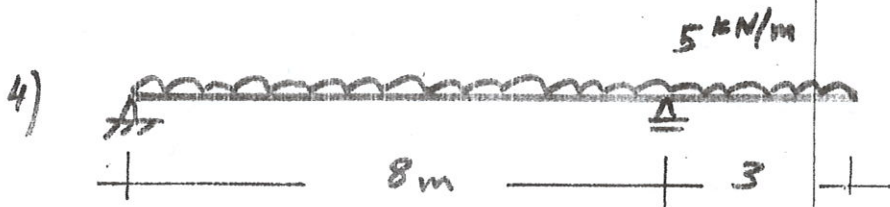
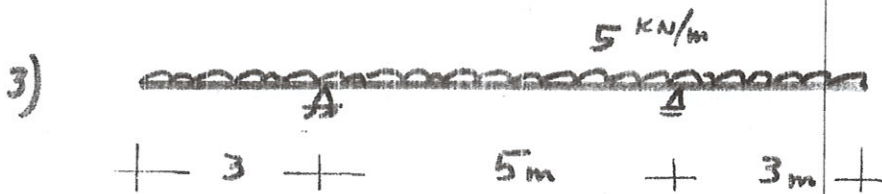
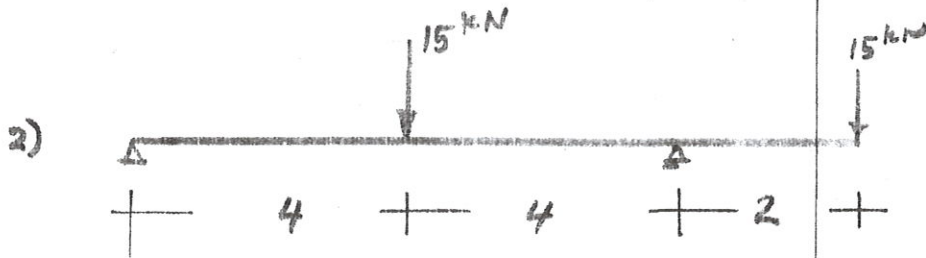
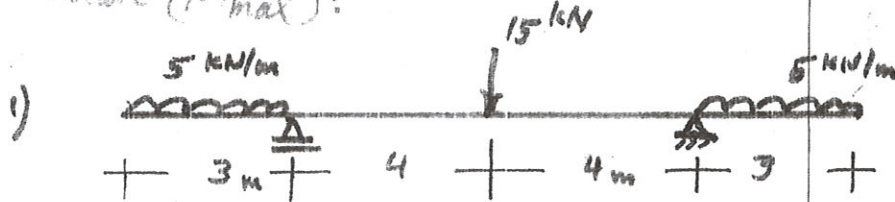
اسم التدريسي: أ.م.د علي العذاري

ع.ب.ب.ب.ب.
ع.ب.ب.ب.ب.

- 9 -
العدد

29

Draw the Bending moment and shearing force for the following
Locate (M_{max}):



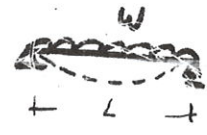
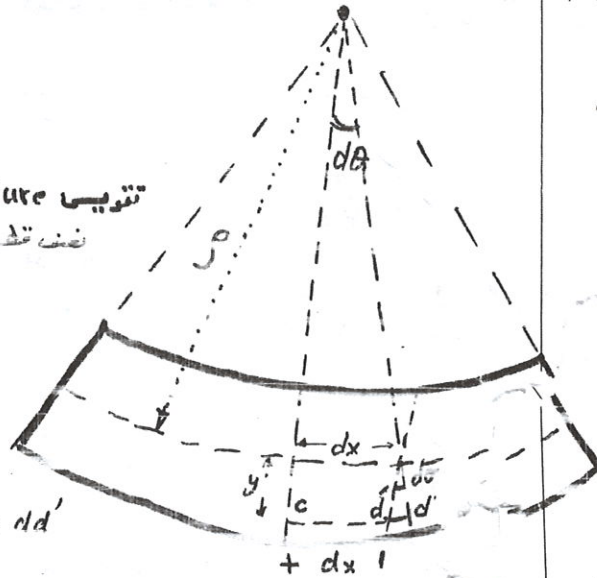


المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د. علي العذاري

Bending stresses in Beams

28-2-2019
 أمومات العزوم في الجسور

تقويس Curvature $\frac{1}{\rho}$
 نصف قطر التقويس



cd' elongate by dd'

$$dd' = y d\theta$$

original length was cd' = dx

$$\epsilon_x = \frac{dd'}{cd'} = \frac{y d\theta}{dx} = y \frac{d\theta}{dx} = \frac{y}{\rho} = \frac{y}{r}$$

$$\epsilon_x = \frac{y}{r}$$

$$d\theta = \frac{dx}{r}$$

$$r = \frac{dx}{d\theta}$$

stress in the Fiber at (y) distance

$$\sigma_x = \epsilon_x \cdot E = \frac{y}{r} \cdot E$$

Force at dA = $\sigma_x \cdot dA$

$$= \frac{y}{r} \cdot E dA$$

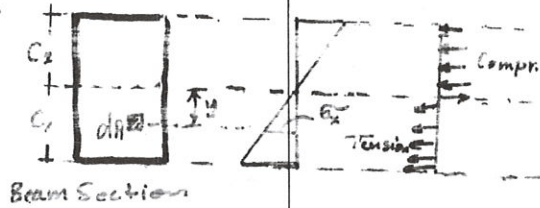
$$= \frac{E}{r} \int y dA$$

M = Force x distance

$$M = \int \sigma_x dA \cdot y = \frac{E}{r} \int y dA \cdot y = \frac{E}{r} \int y^2 dA$$

$$M = \frac{E}{r} \cdot I \quad \frac{M}{EI} = \frac{1}{r}$$

$$\frac{1}{r} = \frac{M}{EI}$$



$$I = \int y^2 dA$$

= moment of Inertia



المرحلة: الثانية
السنة الدراسية: 2017-2018
اسم التدريسي: أ.م.د علي العذاري

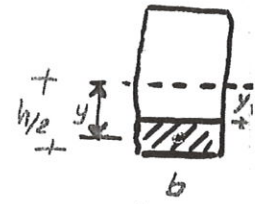
- 2 -

$\int_{y_1}^{c_1} y dA = Q$ مزم المساحة حول المحور المحايد
moment of Area about Neutral axis

$$\tau = \frac{VQ}{Ib}$$

I: moment of Inertia
V: Shearing Force from Shear Force Diagram

(35) (51)



$$Q = \int_{y_1}^{c_1} y dA = \int_{y_1}^{h/2} y b dy = b \int_{y_1}^{h/2} y dy$$

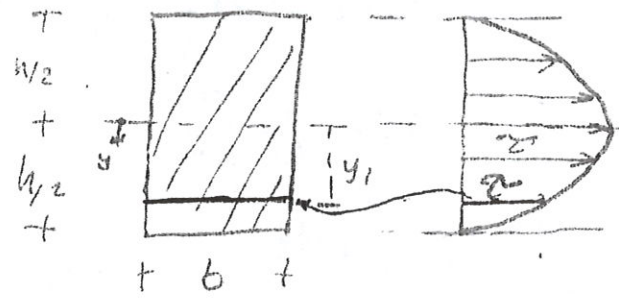
$$Q = b \left(\frac{h}{2} - y_1 \right) \cdot \left[y_1 + \frac{1}{2} \left(\frac{h}{2} - y_1 \right) \right]$$
$$= b \left(\frac{h}{2} - y_1 \right) y_1 + \frac{b}{2} \left(\frac{h}{2} - y_1 \right)^2$$

$$= \frac{b}{2} \left(\frac{h^2}{4} - y_1^2 \right)$$

$$\therefore \tau = \frac{V}{2I} \left(\frac{h^2}{4} - y_1^2 \right)$$

$I = \frac{bh^3}{12}$
V: Shear Force

توزيع قوى القص على المقطع





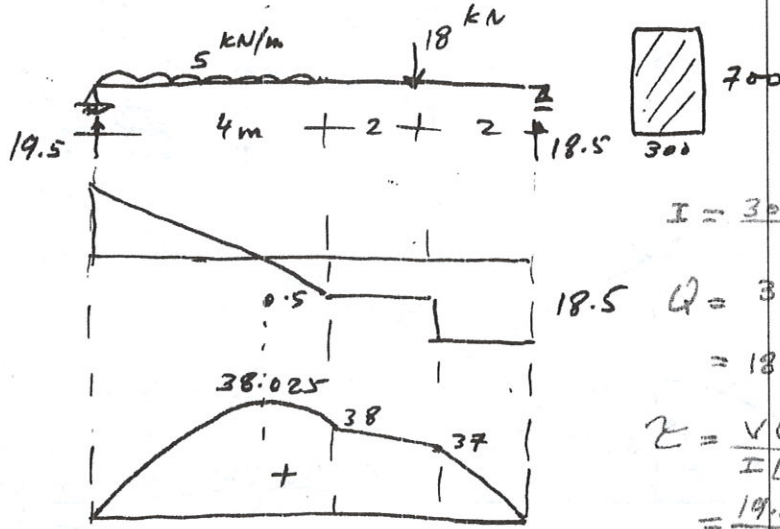
المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د علي العذاري

c-10/2/v

- 3 -

(86) (52)

Example: Calculate the Shearing Stress acting on Beam



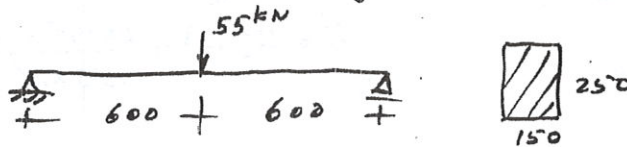
$$I = \frac{300 \times 700^3}{12} = 85.75 \times 10^8 \text{ mm}^4$$

$$Q = 300 \times 350 \times \frac{350}{2} = 18.38 \times 10^6 \text{ mm}^3$$

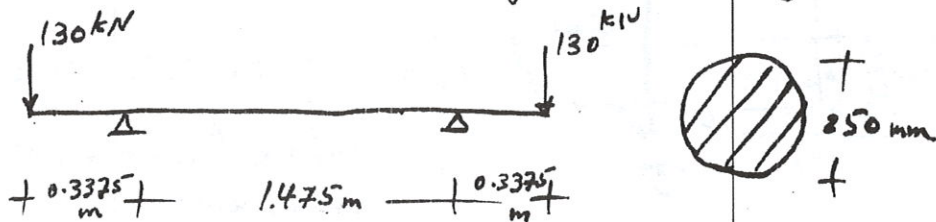
$$\tau = \frac{VQ}{Ib} = \frac{19.5 \times 10^3 \times 18.38 \times 10^6}{85.75 \times 10^8 \times 300} = 0.139 \frac{\text{N}}{\text{mm}^2}$$

واجب

Q1 Calculate the max shearing stresses τ in the Beam :



Q2 - Calculate the max Shearing stresses acting on the Beam



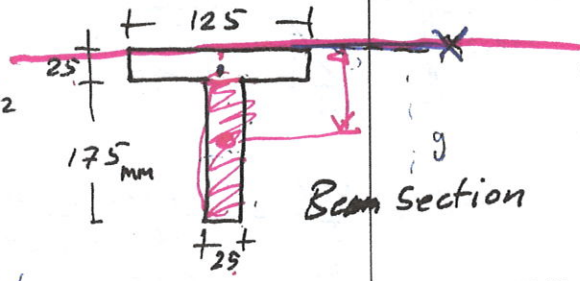
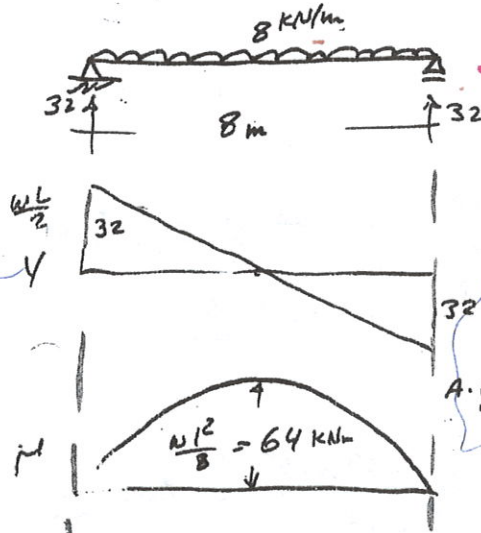
Note: the Centroid of half circle is: $\frac{4r}{3\pi}$
 the moment of Inertia of the circle: $I = \frac{\pi D^4}{64}$



-5-

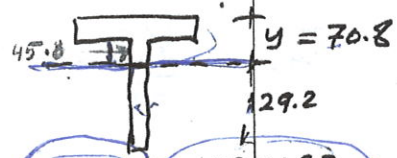
(54)

Example 2: Calculate the $F_{G_{max}}$ and T_{max} for the Beam



$$A = 125 \times 25 + 175 \times 25 = 7500$$

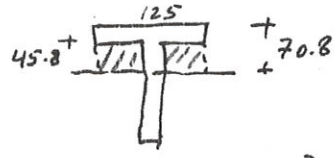
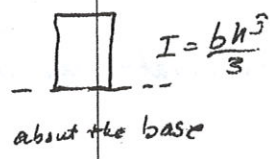
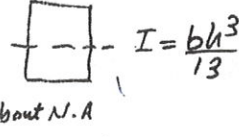
$$A \cdot y = 125 \times 25 \times 12.5 + 175 \times 25 \times 112.5$$



$$\frac{175}{2} + 25 = 112.5$$

$$y = \frac{A_1 y_1 + A_2 y_2}{A_1 + A_2}$$

$$y = \frac{39062 + 492187}{7500} = \frac{531250}{7500} = 70.8 \text{ mm}$$



$$I = \frac{125 \times 70.8^3}{3} + \frac{25 \times 129.2^3}{3} - 2 \times \frac{50 \times 45.8^3}{3}$$

$$I = 14.78 \times 10^6 + 17.97 \times 10^6 - 3 \times 2 \times 10^6 = 29.55 \times 10^6 \text{ mm}^4$$

$$Q = 129.2 \times 25 \times \frac{129.2}{2} = 0.209 \times 10^6 \text{ mm}^3$$

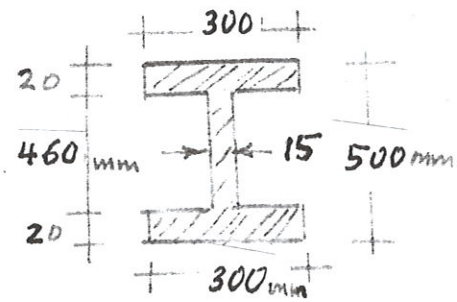
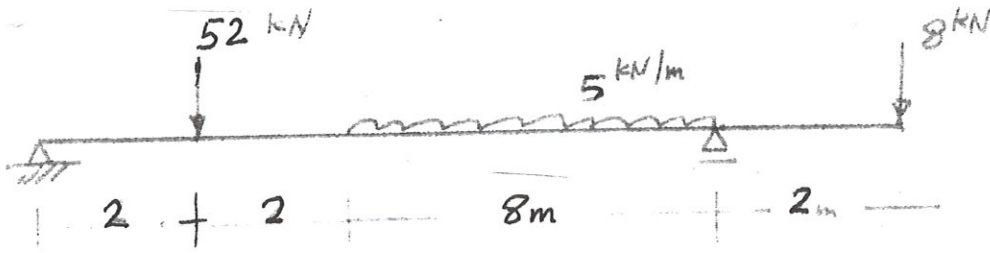


I = _____

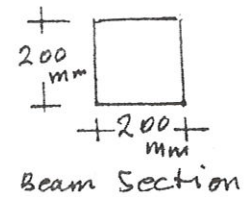
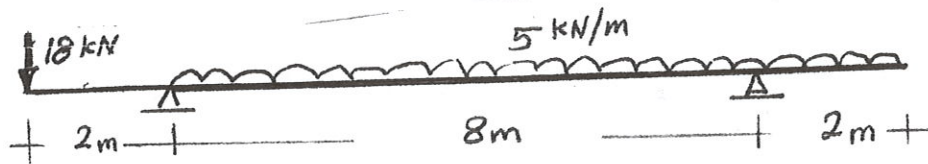
$$y = \frac{A \cdot y}{A}$$

$$A \cdot y = x \cdot y$$

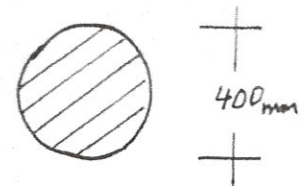
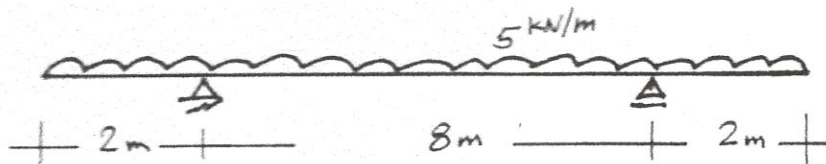
Q2-calculate the Maximum Bending & Shearing STRESSES



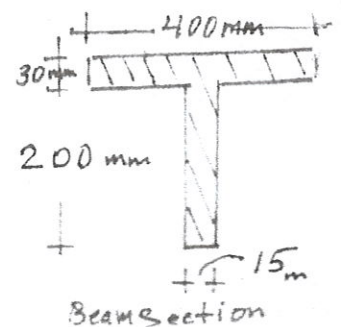
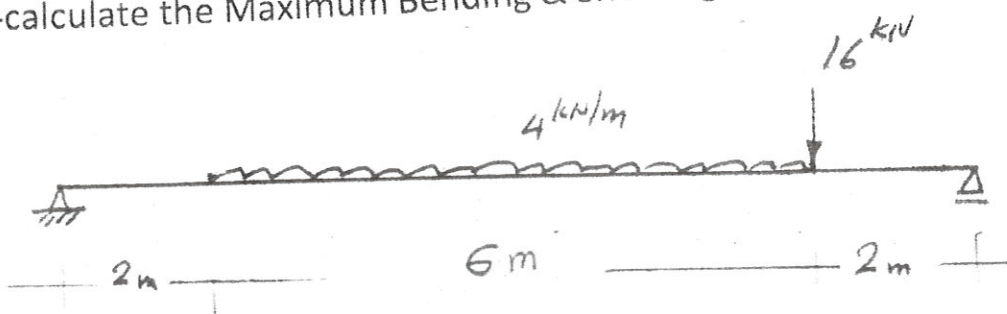
Q1-Draw the Bending & Shearing Force and calculate The stresses



Q1-for the beam calculate the max bending and shearing stresses



Q1-calculate the Maximum Bending & Shearing STRESSES





المرحلة: الثانية
 السنة الدراسية: 2017-2018
 اسم التدريسي: أ.م.د علي العذاري

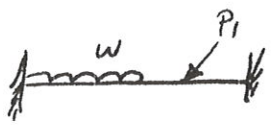
2017/1/4
 2017/2/2
 د. علي العذاري

-1-

Equation of Equilibrium and Compatibility



(37)
 مقاومة المواد
 العصف الثاني / مهدي



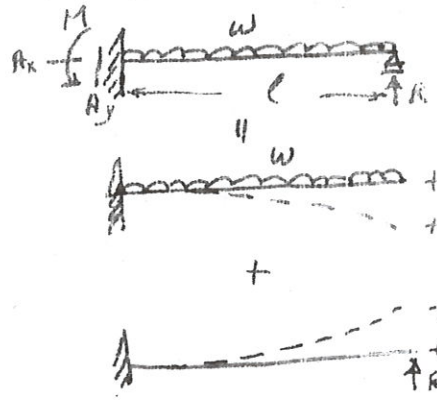
عدد المعاملات	3	5	6
عدد معادلات التوازن	3	3	3
الفائض (Redundant)	0	2	3
عدد المعادلات الإضافية	0	2	3

statically Determinate Problem
 مسائل محددة مسبقاً

statically Indeterminate Problem
 مسائل غير محددة مسبقاً

Compatibility Equations

Example: solve the following problem by computing the Reaction



4 Reactions (statically Indeterminate to the 1st degree) (4-3=1)

We need 1 Compatibility Equation use the deflection condition

$\Delta_1 = \Delta_2 \dots (1)$

$\frac{wL^4}{8EI} = \frac{RL^3}{3EI}$ from table.

$\sum F_x = 0 \dots (2)$

$\sum F_y = 0 \dots (3)$

$\sum M = 0 \dots (4)$

معادلات التوازن
 عدد 3

المرحلة: الثانية

السنة الدراسية: 2017-2018

اسم التدريسي: أ.م.د. علي العذاري



الكلية الإسلامية الجامعة
قسم هندسة تقنيات البناء والإنشاءات
المادة: مقاومة مواد

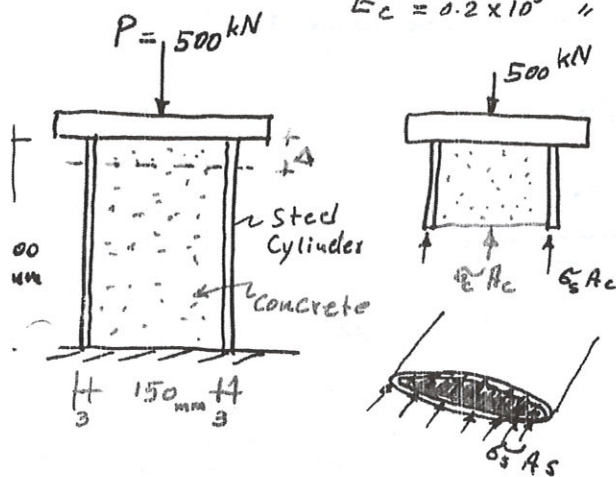
- 2 -

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Example: Calculate the compressive stresses in the steel cylinder and in the concrete, and the total shortening of the cylinder using:

$$E_s = 2 \times 10^5 \text{ MPa} \left[\frac{\text{N}}{\text{mm}^2} \right]$$

$$E_c = 0.2 \times 10^5 \text{ "}$$



Solution:

$$A_s = 2\pi r t = 2\pi \times 75 \times 3 = 1413 \text{ mm}^2$$

$$A_c = \frac{\pi (150)^2}{4} = 17662 \text{ mm}^2$$

2 unknowns مجهولين

$$500 = P_s + P_c \quad \dots (1)$$

التوافق

use compatibility Equation

$$\Delta_c = \Delta_s \quad \dots (2)$$

$$(1) \quad 500 = \sigma_s A_s + \sigma_c A_c$$

$$(2) \quad \dots \quad \frac{P_s \cdot L}{A_s \cdot E_s} = \frac{P_c \cdot L}{A_c \cdot E_c}$$

$$\frac{\sigma_s}{E_s} = \frac{\sigma_c}{E_c} \quad \rightarrow \quad \sigma_s = \frac{E_s}{E_c} \cdot \sigma_c = \frac{2 \times 10^5}{0.2 \times 10^5} \sigma_c$$

$$\sigma_s = 10 \sigma_c$$

$$\dots (2) \quad 500 = \sigma_s A_s + \sigma_c A_c \quad \rightarrow \quad 500 = 10 \sigma_c A_s + \sigma_c A_c$$

$$500 = \sigma_c (10 A_s + A_c)$$

$$\sigma_c = \frac{500}{10 \times 1413 + 17662} =$$

$$\sigma_s = 10 \sigma_c$$

$$P_s = \sigma_s \cdot A_s = 157.2 \times 1413 = 222.12 \text{ kN}$$

$$P_c = \sigma_c \cdot A_c = 15.72 \times 17662 = 277.77$$

$$\left. \begin{array}{l} P_s = 222.12 \text{ kN} \\ P_c = 277.77 \end{array} \right\} \text{المجموع} \rightarrow 500$$

$$\Delta_s = \frac{P_s L}{A_s E_s} = \frac{222.13 \times 300 \times 10^3}{1413 \times 2 \times 10^5} = 0.236 \text{ mm}, \quad \Delta_c = \frac{277.77 \times 10^3 \times 300}{17662 \times 0.2 \times 10^5} = 0.236 \text{ mm}$$

المرحلة: الثانية

السنة الدراسية: 2017-2018

اسم التدريسي: أ.م.د علي العذاري



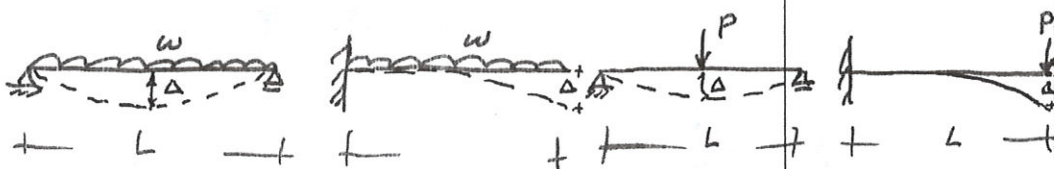
الكلية الإسلامية الجامعة
قسم هندسة تقنيات البناء والانشاءات
المادة: مقاومة مواد

24-4-2015
د.علي العذاري

40
مقاومة المواد
الثاني - مدني

Deflection of Beams

الطبل والشو في الجور

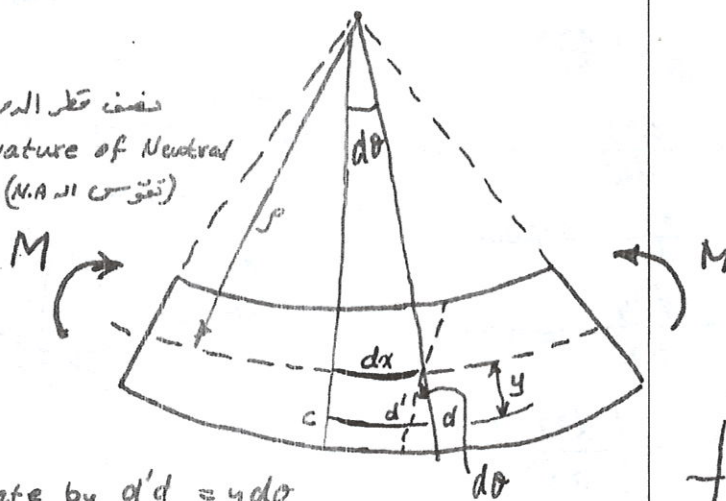


a) Direct Integration method

b) Moment Area method

a) Direct Integration method :

ρ = نصف قطر الدوران
 $\frac{1}{\rho}$ = Curvature of Neutral axis (نقوس ال N.A)



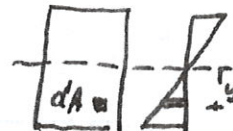
cd' elongate by $d'd = y d\theta$

$$\epsilon_x = \frac{\text{elongation}}{\text{original length}} = \frac{\text{الزيادة في الطول}}{\text{الطول الاصل}} = \frac{y d\theta}{dx}$$

$$\text{من النقوس الكبير} \rightarrow d\theta = \frac{dx}{\rho} \rightarrow \frac{d\theta}{dx} = \frac{1}{\rho}$$

$$\epsilon_x = \frac{y d\theta}{dx} \rightarrow \epsilon_x = \frac{y}{\rho}$$

$$\boxed{\sigma_x = \frac{y}{\rho} E}$$



force and moment

$$dM = \sigma_x dA \cdot y$$

$$M = \int y \sigma_x dA$$

$$\text{for } \sigma_x \quad M = \int_A y \left(\frac{y}{\rho} E \right) dA$$

Cont.

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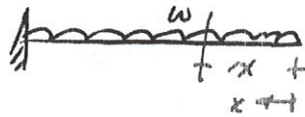


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فلاص

$$y \rightarrow \frac{dy}{dx} = \theta \rightarrow \frac{d^2y}{dx^2} = -M, \frac{d^3y}{dx^3} = -V \rightarrow \frac{d^4y}{dx^4} = w$$

Example 1 Calculate the Deflection at point A for the cantilever



$$M_x = w \cdot x \cdot \frac{x}{2}$$

$$M = -\frac{wx^2}{2}$$

$$dx \times \frac{d^2y}{dx^2} = -\frac{1}{EI} \int wx^2 dx$$

$$1^{st} \text{ deriv. } \frac{dy}{dx} = -\frac{wx^3}{3 \times 2} + C_1$$

for C_1 Boundary Condition
at $x=l$ $\theta = 0$ $y = \phi$

$$\phi = -\frac{wl^3}{6EI} + C_1$$

$$C_1 = \frac{wl^3}{6EI}$$

$$2^{nd} \text{ deriv. at } x=l, y=0 \rightarrow \therefore \frac{dy}{dx} = -\frac{wx^3}{6} + \frac{wl^3}{6EI}$$

$$y = -\frac{wx^4}{24EI} + \frac{wl^3}{6EI} x + C_2$$

$$\phi = -\frac{wl^4}{24EI} + \frac{wl^4}{6EI} + C_2$$

$$\phi = \frac{wl^4(-1+4)}{24} + C_2$$

$$C_2 = \frac{-3wl^4}{24EI} \rightarrow C_2 = -\frac{wl^4}{8}$$

$$y = \frac{wx^4}{24EI} + \frac{wl^3}{6EI} x - \frac{wl^4}{8EI}$$

at point A $x=0$

$$y_0 = \frac{-wl^4}{8EI}$$



we have the Equation:

$$\frac{d^2y}{dx^2} = -\frac{M}{EI}$$

1st derivative gives θ
2nd " " " y

الشروط الحدية

في الحرة

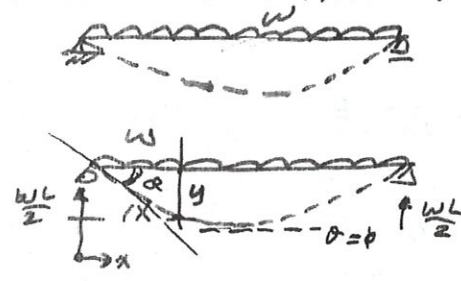


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(44)

V

Example 3: Calculate the max deflection



$$M = \frac{wL}{2}x - \frac{wx^2}{2}$$

$$EI \frac{d^2y}{dx^2} = -M$$

$$dx \times EI \frac{d^2y}{dx^2} = \frac{wL}{2}x - \frac{wx^2}{2}$$

$$\int EI \frac{d^2y}{dx^2} = \int (\frac{wL}{2}x - \frac{wx^2}{2}) dx$$

$$EI \frac{dy}{dx} = (\frac{wL}{2} \frac{x^2}{2} - \frac{wx^3}{2 \times 3}) + C_1$$

for C_1 Boundary condition at $x = \frac{L}{2} \rightarrow \frac{dy}{dx} = 0$ ($\theta = \phi$)

$$0 = \frac{wL}{2} \frac{1}{2} \frac{L^2}{4} - \frac{w}{6} \frac{L^3}{8} + C_1$$

$$0 = \frac{wL^3}{16} - \frac{wL^3}{48} + C_1$$

$$C_1 = \frac{-wL^3}{24}$$

$$\therefore \int EI \frac{dy}{dx} = \int (\frac{wL}{4}x^2 - \frac{wx^3}{6} - \frac{wL^3}{24})$$

$$EI y = \frac{wL}{4} \frac{x^3}{3} - \frac{wx^4}{6 \times 4} - \frac{wL^3}{24} x + C_2$$

for C_2 : at $x = 0$ $y = 0$

$$0 = 0 - 0 + 0 + C_2 \rightarrow C_2 = 0$$

$$\therefore EI y = \frac{wL}{12} x^3 - \frac{wx^4}{24} - \frac{wL^3}{24} x$$

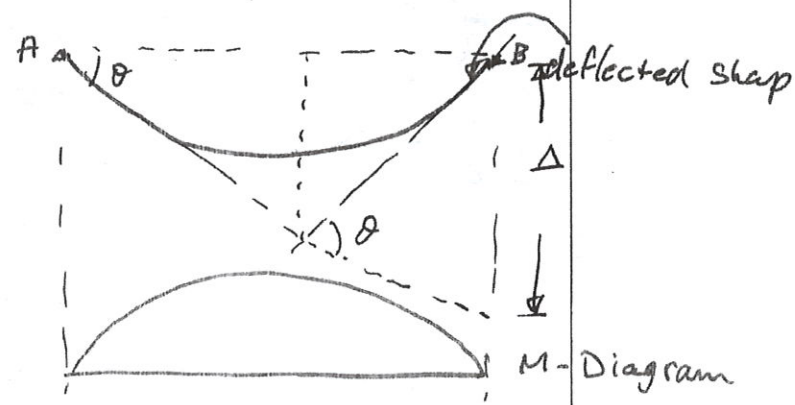
VI



(46)

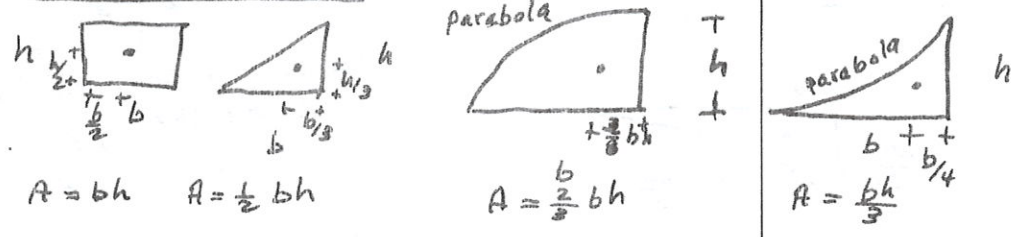
A

b) Moment-area Method

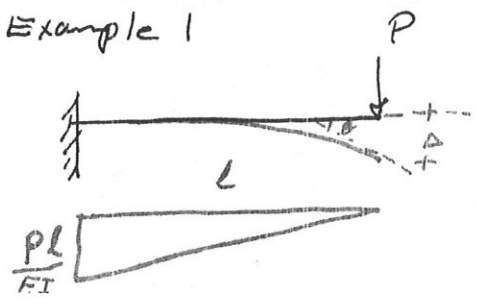


- 1) The Angle between two tangent = The Area of the Moment Diagram $\times \frac{1}{EI}$
- 2) The Vertical displacement A and B = The moment of the $\frac{M}{EI}$ diagram about B

useful Area Data :



Example 1



زاوية العزم بين النقطتين =

$$= \frac{1}{2} \times \frac{PL}{EI} \cdot L = \frac{PL^2}{2EI}$$

عزم العزم بين النقطتين حول الافرجه =

$$\Delta = \frac{1}{2} \times \frac{PL}{EI} \cdot L \cdot \frac{2}{3} L = \frac{PL^3}{3EI}$$

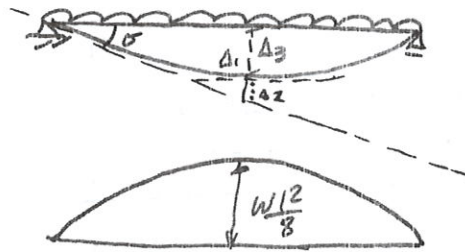


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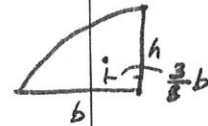
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Example 4



(48)

من الجدول:



$$A = \frac{2}{3} bh$$

$$\text{عزم المساحة} = \text{التأثير} = A = \frac{2}{3} \frac{wL^2}{8} \times \frac{1}{2} = \frac{wL^3}{24EI}$$

$$\Delta_1 = \theta \cdot \frac{L}{2} = \frac{wL^4}{48EI}$$

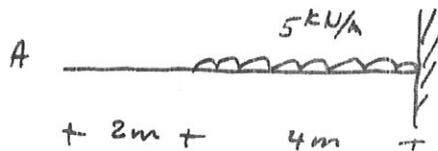
$$\text{عزم المساحة} = \text{البعد بين المحاور} = \Delta_2 = \frac{wL^3}{24EI} \times \frac{3}{8} \cdot \frac{L}{2} = \frac{wL^4}{128}$$

$$\begin{aligned} \Delta_3 &= \Delta_1 - \Delta_2 \\ &= \frac{wL^4}{48EI} - \frac{wL^4}{128EI} = \frac{wL^4}{16EI} \left(\frac{1}{3} - \frac{1}{8} \right) \\ &= \frac{wL^4}{16EI} \cdot \frac{5}{24} \end{aligned}$$

$$\boxed{\Delta_3 = \frac{5}{384} wL^4}$$

واجب

Calculate the vertical deflection on point A



$$\text{Answer: } \frac{266}{EI}$$



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15/11/20
 د. علي العذاري

Pressure Vessels

أواني الضغط

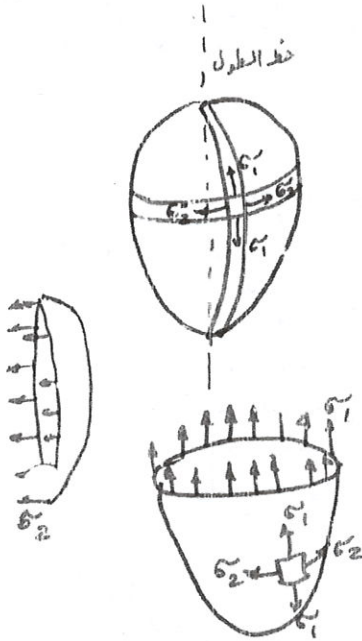
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ميرضا: $\frac{\text{القوة}}{\text{المساحة}} = \frac{P}{A} = \sigma$ الإبراد

القوة = الإبراد × المساحة
 $A \times \sigma = P$

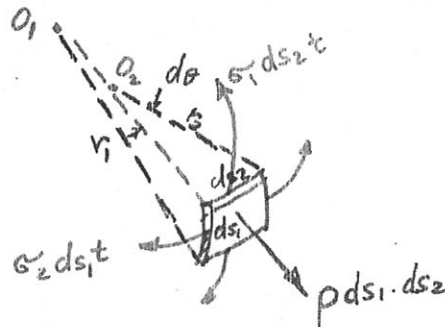
(4)

- σ_1 = قوة الشد باتجاه خط الطول meridional stress
- σ_2 = قوة الشد باتجاه القطر hoop stress
- t = سمك الاناء
- r_1 = القطر باتجاه خط الطول
- r_2 = القطر باتجاه محوري خط الطول

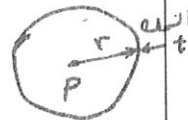


نتيجة الاشتقاق

$$\frac{\sigma_1}{r_1} + \frac{\sigma_2}{r_2} = \frac{P}{t}$$



Example 1: Calculate the membrane stress σ_1, σ_2 for the sphere with Press. (P)



الحل solution: $\frac{\sigma_1}{r_1} + \frac{\sigma_2}{r_2} = \frac{P}{t}$

$r = r_2 = r_1$ (نصف الكرة)
 $\sigma = \sigma_2 = \sigma_1$

$\frac{\sigma}{r} + \frac{\sigma}{r} = \frac{P}{t}$
 $\frac{2\sigma}{r} = \frac{P}{t} \rightarrow \sigma = \frac{Pr}{2t}$



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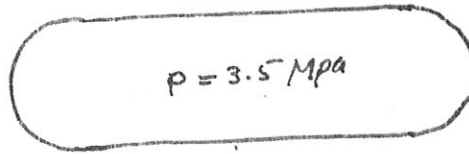
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3
 Pressure Vessels (Problems)

مقاومة المواد
 (3)

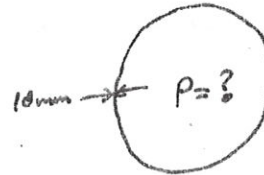
Solve the following Equations:

Q₁ - Calculate the safe thickness t for the cylindrical pressure vessel having $\sigma_t = 250 \text{ MPa}$ in order to resist an internal pressure $P = 3.5 \text{ MPa}$ using a safety factor of 3.5?



+
 600 mm
 +

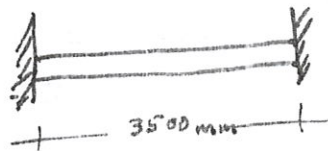
Q₂ - A spherical Tank having $D = 2000 \text{ mm}$ and $t = 10 \text{ mm}$ made from material having $\sigma = 125 \text{ MPa}$. What is the safe Gass pressure it can resist?



+
 2000 mm
 +

Temperature change

Q₃ - A copper bar shown. Calculate the stresses when the Temperature drop from 85°C to 50°C using:



$$\alpha = 0.000018 \frac{1}{^\circ\text{C}}$$

$$E = 1.2 \times 10^3 \frac{\text{N}}{\text{mm}^2}$$

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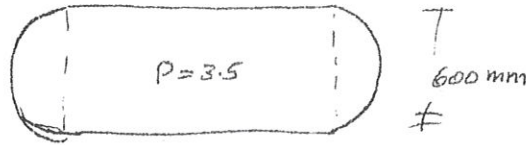
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Example

$$\frac{N}{mm^2} = \frac{MN}{m^2} = MPa$$



Calculate the safe thickness t for the cylinder having $\sigma_t = 25$ MPa
in order to resist an internal pressure $P = 3.5$ MPa ($\frac{N}{mm^2}$)
using a safety factor of 3.5

$$\frac{\sigma_1}{r_1} + \frac{\sigma_2}{r_2} = \frac{P}{r}$$

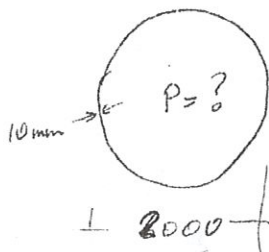
$$\sigma_1 = \frac{Pr}{2t}$$

$$\sigma_2 = \frac{Pr}{t} \rightarrow \text{Critical}$$

$$\sigma_2 = \frac{Pr}{t} \quad 7.35$$

$$t = \frac{3.5 \times \left(\frac{600}{2}\right)}{\frac{25}{3.5}} = 14.7 \text{ mm}$$

Spherical Tank having $D = 2000$ mm and $t = 10$ mm and
 $\sigma_t = 125$ MPa ($\frac{N}{mm^2}$) what is the safe Gas pressure

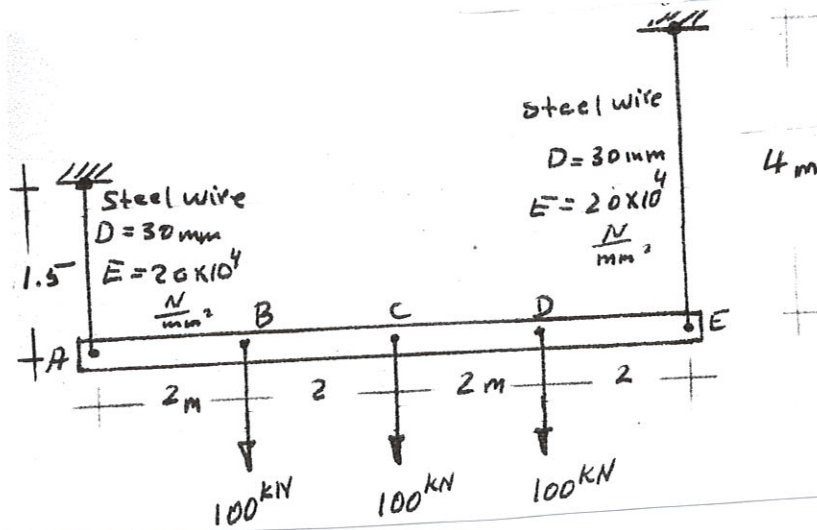


$$\sigma = \frac{Pr}{2t}$$

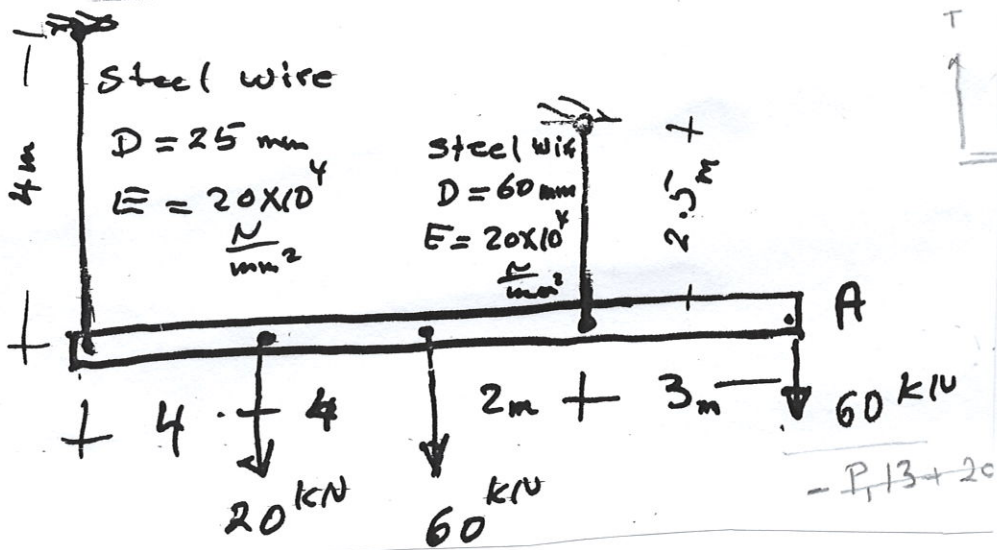
$$125 = \frac{P \times 1000}{2 \times 10}$$

$$P = \frac{125 \times 2}{100} = \frac{2500}{100} = 25 \text{ MPa}$$

Q1- calculate the vertical movement of points A, B, C, D and E



Q2 calculate the Vertical movement of point A

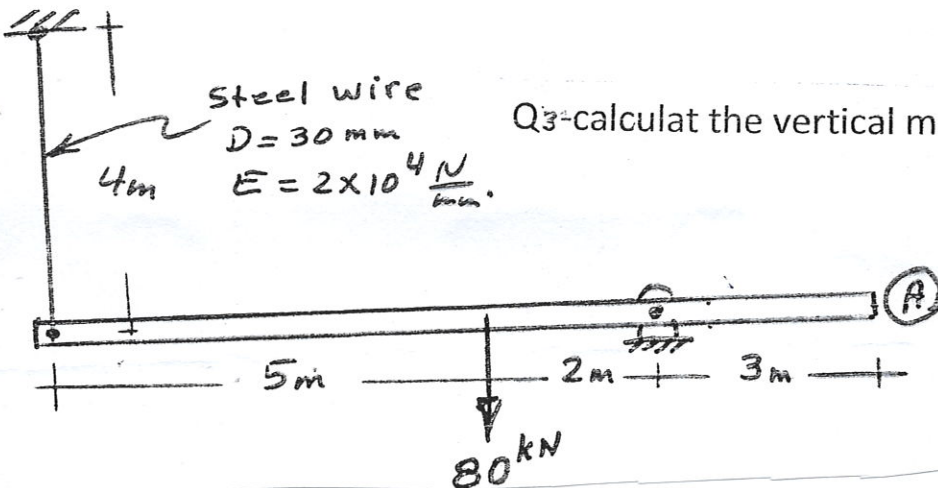


$\uparrow \downarrow = 1$
 $20 + 60$

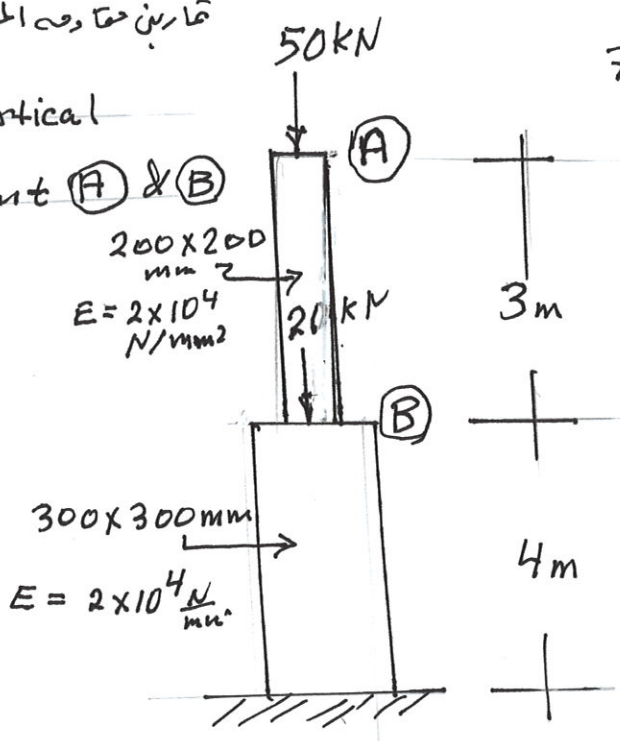
\uparrow

$- P_1 + 20$

Q3- calculate the vertical movement of Point A.

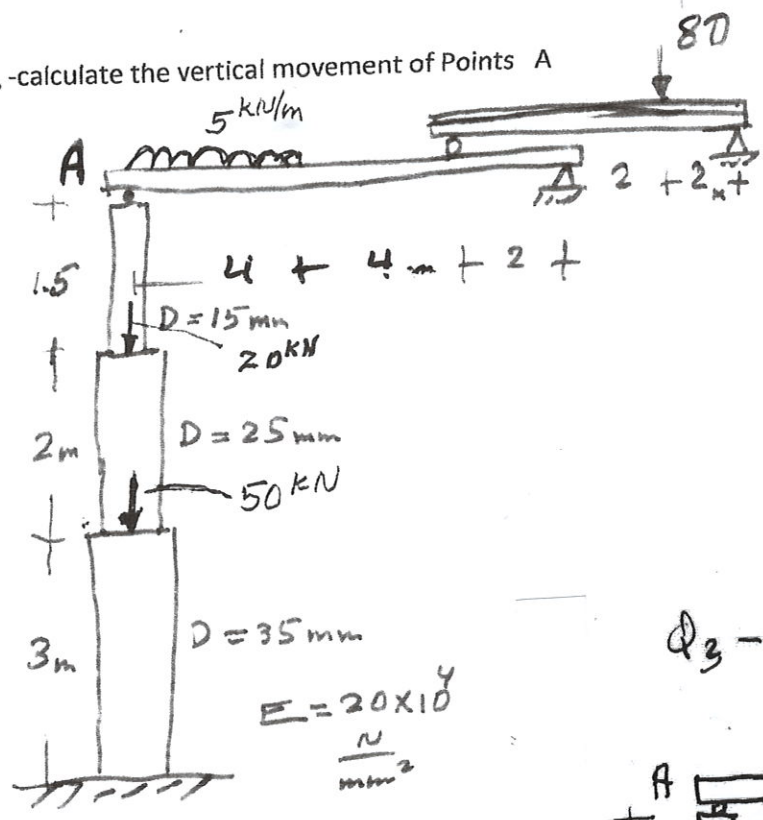


Q1 - Calculate the vertical movement of point (A) & (B)



0,33

Q2 - calculate the vertical movement of Points A



Q3 - Calculate $\Delta V_A = ?$

